

1.0

EGYSZABADSAÁGFOKÚ REZGŐRENDSZER MOZGÁSEGYENLETE

• általános koordináták

$$q = y_B \quad - B \text{ pont elmozdulása}$$

$$\dot{q} = \dot{y}_B \quad (v_B) \quad - B \text{ pont sebessége}$$

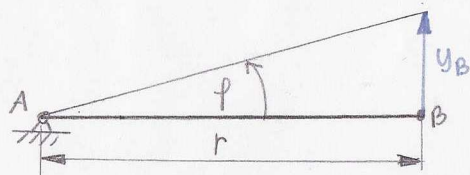
$$\ddot{q} = \ddot{y}_B \quad (a_B) \quad - B \text{ pont gyorsulása}$$

$$q = \varphi \quad \text{szögelfordulás}$$

$$\dot{q} = \dot{\varphi} \quad (\omega) \quad \text{szögsebesség}$$

$$\ddot{q} = \ddot{\varphi} \quad (\varepsilon) \quad \text{szöggyorsulás}$$

• összefüggés általános koordináták között



$$y_B = r \cdot \varphi$$

$$\dot{y}_B = r \cdot \dot{\varphi}$$

$$\ddot{y}_B = r \cdot \ddot{\varphi}$$

• Lagrange egyenlet: $\frac{d}{dt} \left(\frac{\partial E}{\partial \dot{q}} \right) - \frac{\partial E}{\partial q} = Q_c + Q_k + Q_g$

1. $\frac{d}{dt} \left(\frac{\partial E}{\partial \dot{q}} \right)$ meghatározása

1. lépés: rendszer mozgási energiája

a) tömegpont: $E = \frac{1}{2} m v^2$

$$\dot{q} = \dot{y}_B \quad \frac{1}{2} m \dot{y}_B^2$$

$$\dot{q} = \dot{\varphi} \quad \frac{1}{2} m (r \cdot \dot{\varphi})^2$$

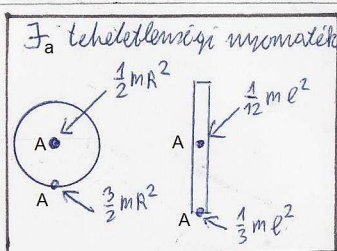
b) merev test: $E = \frac{1}{2} J_a \omega^2$

$$\frac{1}{2} J_a \left(\frac{\dot{y}_B}{r} \right)^2$$

$$\frac{1}{2} J_a \dot{\varphi}^2$$

az energia általános alakja: $E = \frac{1}{2} \cdot m_{red} \cdot \dot{q}^2$

2. lépés: deriválás: $\frac{d}{dt} \left(\frac{\partial E}{\partial \dot{q}} \right) = m_{red} \cdot \ddot{q}$



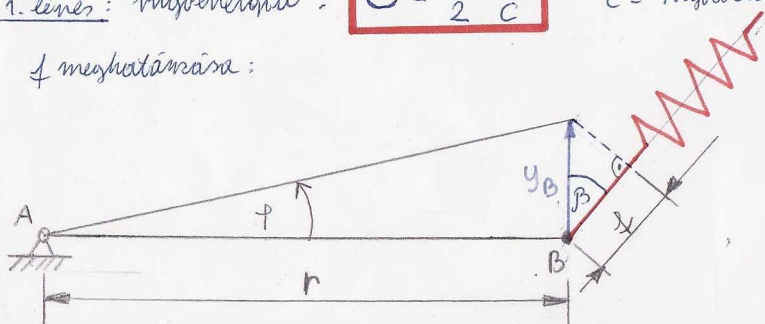
2. $\frac{\partial E}{\partial q}$ meghatározása

$$\frac{\partial E}{\partial q} = 0$$

3. Q_c meghatározása

1. lépés: rugóenergia: $U = \frac{1}{2} \frac{f^2}{c}$

f meghatározása:



$$f = y_B \cdot \cos \beta$$

$$y_B = r \cdot \varphi$$

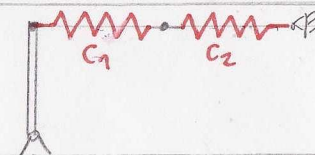
$$f = r \cdot \varphi \cdot \cos \beta$$

2. lépés: rugóerő

$$Q_c = - \frac{\partial U}{\partial q} = - \frac{1}{c_{red}} \cdot q$$

szoba kapcsolt rugók:

$$U = \frac{1}{2} \cdot \frac{f^2}{c_1 + c_2}$$



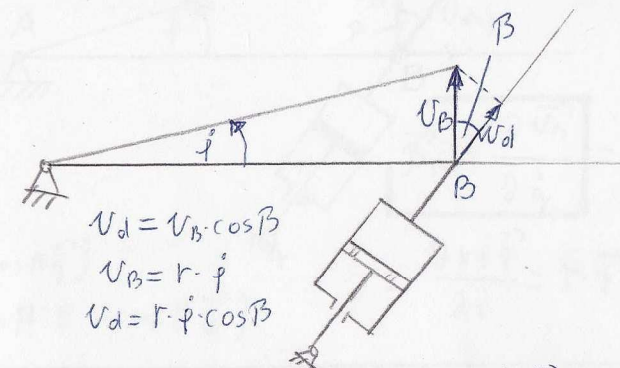
4. Q_k meghatározása

1. lépés: villanytárolási energia

$$D = \frac{1}{2} k v_d^2$$

2. lépés: villanytároló erő

$$Q_k = - \frac{\partial D}{\partial \dot{q}} = - k_{red} \cdot \dot{q}$$



$$v_d = v_B \cdot \cos \beta$$

$$v_B = r \cdot \dot{\varphi}$$

$$v_d = r \cdot \dot{\varphi} \cdot \cos \beta$$

5. Q_g meghatározása

$$Q_g = \vec{F}_g(t) \cdot \vec{\beta}_B$$

$$\text{vagy } Q_g = \vec{M}_g(t) \cdot \vec{b}$$

$$\vec{\beta}_B = \frac{\partial \vec{r}_B}{\partial \dot{q}}$$

$$\vec{b} = \frac{\partial \vec{r}}{\partial \dot{q}}$$

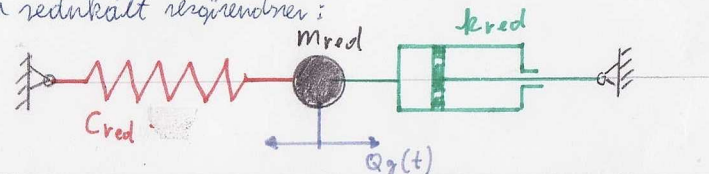
• lehetetlenítés Lagrange egyenletre

$$m_{red} \cdot \ddot{q} - 0 = - \frac{1}{c_{red}} \cdot q - k_{red} \cdot \dot{q} + Q_g(t)$$

• rendszer az egyenletet:

$$m_{red} \ddot{q} + k_{red} \dot{q} + \frac{1}{c_{red}} q = Q_g(t)$$

• a redukált rezgőrendszer:



• általános koordináták

$q = y_B$ - B pont elmozdulása

$\dot{q} = \dot{y}_B$ (v_B) - B pont sebessége

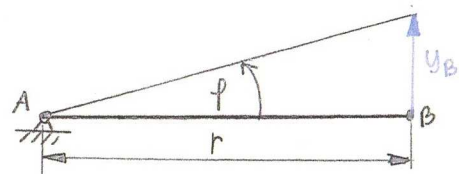
$\ddot{q} = \ddot{y}_B$ (a_B) - B pont gyorsulása

$q = \varphi$ - szögelfordulás

$\dot{q} = \dot{\varphi}$ (ω) - szögsebesség

$\ddot{q} = \ddot{\varphi}$ (ϵ) - szöggyorsulás

• összefüggés általános koordináták között



$$y_B = r \cdot \varphi$$

$$\dot{y}_B = r \cdot \dot{\varphi}$$

$$\ddot{y}_B = r \cdot \ddot{\varphi}$$

• Lagrange egyenlet: $\frac{d}{dt} \left(\frac{\partial E}{\partial \dot{q}} \right) - \frac{\partial E}{\partial q} = Q_c + Q_k + Q_g$

1. $\frac{d}{dt} \left(\frac{\partial E}{\partial \dot{q}} \right)$ meghatározása

1. lépés: rendszer összes energiája

	$\dot{q} = \dot{y}_B$	$\dot{q} = \dot{\varphi}$
a) tömegpont: $E = \frac{1}{2} m v^2$	$\frac{1}{2} m \dot{y}_B^2$	$\frac{1}{2} m (r \dot{\varphi})^2$
b) merev test: $E = \frac{1}{2} J_a \omega^2$	$\frac{1}{2} J_a \left(\frac{\dot{y}_B}{r} \right)^2$	$\frac{1}{2} J_a \dot{\varphi}^2$

az energia általános alakja: $E = \frac{1}{2} \cdot m_{red} \cdot \dot{q}^2$

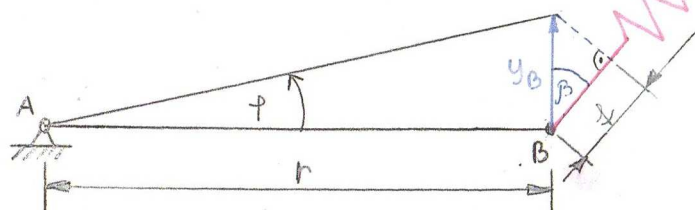
2. lépés: deriválás: $\frac{d}{dt} \left(\frac{\partial E}{\partial \dot{q}} \right) = m_{red} \cdot \ddot{q}$

2. $\frac{\partial E}{\partial q}$ meghatározása $\frac{\partial E}{\partial q} = 0$

3. Q_c meghatározása

1. lépés: rugóenergia: $U = \frac{1}{2} \frac{f^2}{c}$

f meghatározása:



$$f = y_B \cdot \cos \beta$$

$$y_B = r \cdot \varphi$$

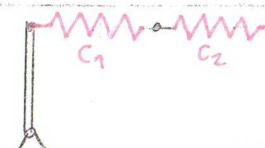
$$f = r \cdot \varphi \cdot \cos \beta$$

2. lépés: nyújtás

$$Q_c = - \frac{\partial U}{\partial q} = - \frac{1}{c_{red}} \cdot q$$

széria kapcsolt rugók:

$$U = \frac{1}{2} \cdot \frac{f^2}{c_1 + c_2}$$



4. Q_k meghatározása

$$Q_k = \vec{F}_k \cdot \vec{\beta}_B$$

$$\vec{F}_k = -k \cdot \vec{v}_{dvug}$$

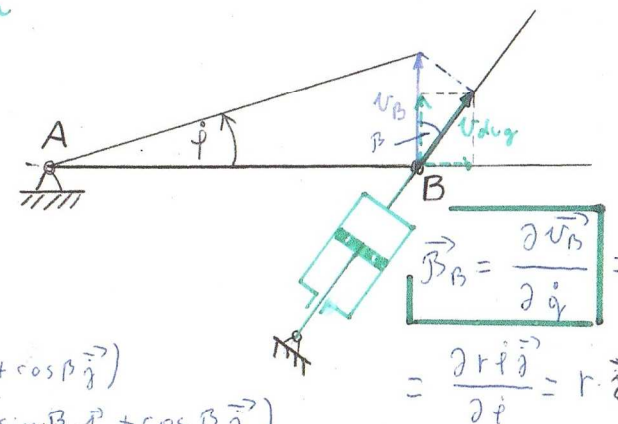
$$v_{dvug} = v_B \cdot \cos \beta$$

$$v_B = r \cdot \dot{\varphi}$$

$$v_{dvug} = r \cdot \dot{\varphi} \cdot \cos \beta$$

$$\vec{v}_{dvug} = v_{dvug} (\sin \beta \vec{i} + \cos \beta \vec{j})$$

$$\vec{v}_{dvug} = r \cdot \dot{\varphi} \cdot \cos \beta \cdot (\sin \beta \vec{i} + \cos \beta \vec{j})$$



$$\vec{\beta}_B = \frac{\partial \vec{v}_B}{\partial \dot{q}} = \frac{\partial (r \dot{\varphi} \vec{j})}{\partial \dot{\varphi}} = r \cdot \vec{j}$$

5. Q_g meghatározása

$$Q_g = \vec{F}_g(t) \cdot \vec{\beta}_B$$

$$\text{vagy } Q_g = \vec{M}_g(t) \cdot \vec{b}$$

$$\vec{\beta}_B = \frac{\partial \vec{v}_B}{\partial \dot{q}} \quad \vec{b} = \frac{\partial \vec{\omega}}{\partial \dot{q}}$$

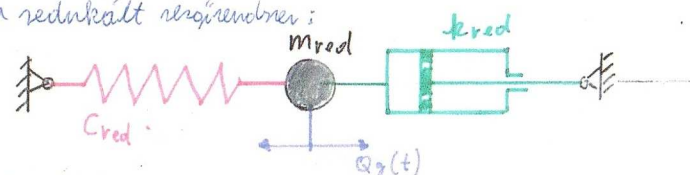
• leheletlenlis Lagrange egyenletbe

$$m_{red} \cdot \ddot{q} - 0 = - \frac{1}{c_{red}} \cdot q - k_{red} \cdot \dot{q} + Q_g(t)$$

• rendszer ar egyenletet:

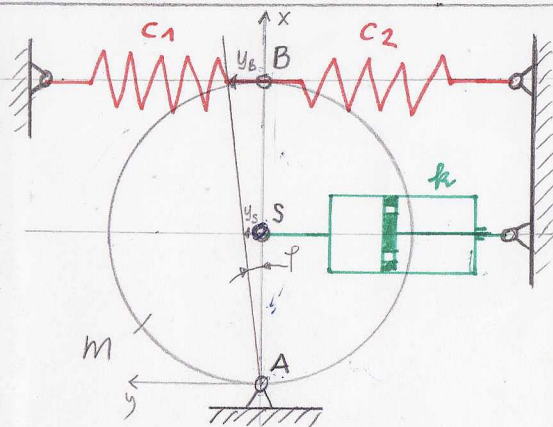
$$m_{red} \ddot{q} + k_{red} \dot{q} + \frac{1}{c_{red}} q = Q_g(t)$$

• a redukált rendszerben:



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EGYSZABADSÁGFOKÚ REZGŐRENDSZER MOZGÁSEGYENLETE



adott: c_1, c_2, k, m, R

• az általános koordináta:

a) $q = \varphi$ b) $q = y_B$
 $\dot{q} = \dot{\varphi}(\omega)$ $\dot{q} = \dot{y}_B (v_B)$
 $\ddot{q} = \ddot{\varphi}(\varepsilon)$ $\ddot{q} = \ddot{y}_B (a_B)$

• Lagrange egyenlet:

$$\frac{d}{dt} \left(\frac{\partial E}{\partial \dot{q}} \right) - \frac{\partial E}{\partial q} = Q_c + Q_k$$

$$= - \left(\frac{4R^2}{c_1} + \frac{4R^2}{c_2} \right) \cdot \ddot{\varphi} = - \left(\frac{4R^2}{c_1} + \frac{4R^2}{c_2} \right) \cdot \ddot{\varphi}$$

$$Q_c = - \left(\frac{4R^2}{c_1} + \frac{4R^2}{c_2} \right) \cdot \ddot{\varphi}$$

4. Q_k meghatározása

$$D = \frac{1}{2} k v_d^2 = \frac{1}{2} k (R \cdot \dot{\varphi})^2 = \frac{1}{2} k \cdot R^2 \dot{\varphi}^2$$

$$Q_k = - \frac{\partial D}{\partial \dot{\varphi}} = - k R^2 \dot{\varphi}$$

$$Q_k = - k R^2 \dot{\varphi}$$

• lehelyettesítés Lagrange egyenletbe

$$\frac{3}{2} m R^2 \ddot{\varphi} - 0 = - \left(\frac{4R^2}{c_1} + \frac{4R^2}{c_2} \right) \cdot \ddot{\varphi} - k R^2 \dot{\varphi}$$

• rendezés az egyenletet:

$$\underbrace{\frac{3}{2} m R^2 \cdot \ddot{\varphi}}_{m_{red}} + \underbrace{k \cdot R^2 \cdot \dot{\varphi}}_{k_{red}} + \underbrace{\left(\frac{4R^2}{c_1} + \frac{4R^2}{c_2} \right) \cdot \ddot{\varphi}}_{\frac{1}{c_{red}}} = 0$$

• a redukált rezgőrendszer:



$$m_{red} \cdot \ddot{q} + k_{red} \cdot \dot{q} + \frac{1}{c_{red}} \cdot q = 0$$

b) 1. $\frac{d}{dt} \left(\frac{\partial E}{\partial \dot{q}} \right)$ meghatározása

$$E = \frac{1}{2} J_a \omega^2 = \frac{1}{2} \left(\frac{3}{2} m R^2 \right) \cdot \omega^2 = \frac{1}{2} \left(\frac{3}{2} m R^2 \right) \cdot \left(\frac{\dot{y}_B}{2R} \right)^2 = \frac{1}{2} \cdot \left(\frac{3}{8} m \right) \cdot \dot{y}_B^2$$

$$\frac{\partial E}{\partial \dot{y}_B} = \frac{3}{8} m \dot{y}_B$$

$$\frac{d}{dt} \left(\frac{\partial E}{\partial \dot{y}_B} \right) = \frac{3}{8} m \ddot{y}_B$$

$$\frac{d}{dt} \left(\frac{\partial E}{\partial \dot{q}} \right) = \frac{3}{8} m \ddot{y}_B$$

$$2. \frac{\partial E}{\partial q} = 0$$

3. Q_c meghatározása

$$U = U_1 + U_2 = \frac{1}{2} \frac{y_B^2}{c_1} + \frac{1}{2} \frac{y_B^2}{c_2} = \frac{1}{2} \cdot \left(\frac{1}{c_1} + \frac{1}{c_2} \right) y_B^2$$

$$Q_c = - \frac{\partial U}{\partial y_B} = - \left(\frac{1}{c_1} + \frac{1}{c_2} \right) y_B$$

$$Q_c = - \left(\frac{1}{c_1} + \frac{1}{c_2} \right) y_B$$

4. Q_k meghatározása

$$D = \frac{1}{2} k v_d^2 = \frac{1}{2} k \cdot \left(\frac{\dot{y}_B}{2} \right)^2 = \frac{1}{2} k \cdot \frac{1}{4} \cdot \dot{y}_B^2$$

$$Q_k = - \frac{\partial D}{\partial \dot{y}_B} = - k \cdot \frac{1}{4} \cdot \dot{y}_B$$

$$Q_k = - k \cdot \frac{1}{4} \cdot \dot{y}_B$$

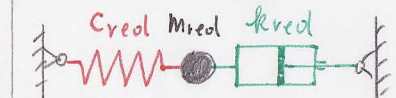
• lehelyettesítés Lagrange egyenletbe

$$\frac{3}{8} m \ddot{y}_B - 0 = - \left(\frac{1}{c_1} + \frac{1}{c_2} \right) y_B - \frac{k}{4} \cdot \dot{y}_B$$

• rendezés az egyenletet

$$\underbrace{\frac{3}{8} m \ddot{y}_B}_{m_{red}} + \underbrace{\frac{k}{4} \cdot \dot{y}_B}_{k_{red}} + \underbrace{\left(\frac{1}{c_1} + \frac{1}{c_2} \right) y_B}_{\frac{1}{c_{red}}} = 0$$

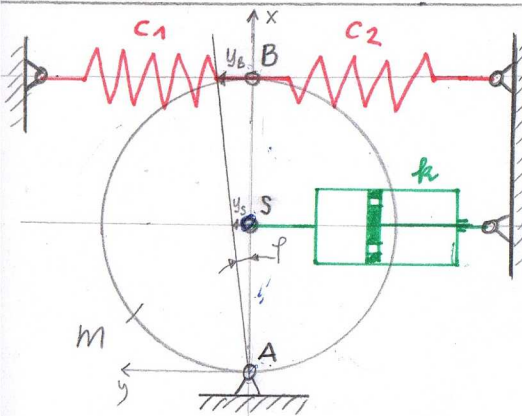
• a redukált rezgőrendszer:



$$m_{red} \ddot{q} + k_{red} \cdot \dot{q} + \frac{1}{c_{red}} \cdot q = 0$$

1.1

EGYSZABADSÁGFOKÚ REZGŐRENDSZER MOZGÁSEGYENLETE



adott: c_1, c_2, k, m, R

• az általános koordináta:

$$\begin{aligned} \text{a) } q &= \varphi & \text{b) } q &= y_B \\ \dot{q} &= \dot{\varphi}(\omega) & \dot{q} &= \dot{y}_B(v_B) \\ \ddot{q} &= \ddot{\varphi}(\epsilon) & \ddot{q} &= \ddot{y}_B(a_B) \end{aligned}$$

• Lagrange egyenlet:

$$\frac{d}{dt} \left(\frac{\partial E}{\partial \dot{q}} \right) - \frac{\partial E}{\partial q} = Q_c + Q_k$$

$$= - \left(\frac{4R^2}{c_1} + \frac{4R^2}{c_2} \right) \cdot \varphi$$

$$Q_c = - \left(\frac{4R^2}{c_1} + \frac{4R^2}{c_2} \right) \cdot \varphi$$

④ Q_k meghatározása

$$Q_k = \vec{F}_k \cdot \vec{\beta}_s$$

$$\vec{F}_k = -k \cdot \vec{v}_{\text{ang}} = -k \cdot \vec{v}_s = -k \cdot (R \cdot \dot{\varphi}) \vec{j}$$

$$\vec{\beta}_s = \frac{\partial \vec{v}_s}{\partial \dot{\varphi}} = \frac{\partial (R \cdot \dot{\varphi})}{\partial \dot{\varphi}} = R \cdot \vec{j}$$

$$Q_k = -k \cdot R \cdot \dot{\varphi} \cdot \vec{j} \cdot R \cdot \vec{j} = -k R^2 \dot{\varphi}$$

$$Q_k = -k R^2 \dot{\varphi}$$

• lehelyettesítés Lagrange egyenletbe

$$\frac{3}{2} m R^2 \ddot{\varphi} - 0 = - \left(\frac{4R^2}{c_1} + \frac{4R^2}{c_2} \right) \cdot \varphi - k R^2 \dot{\varphi}$$

• rendezés az egyenletet:

$$\underbrace{\frac{3}{2} m R^2 \ddot{\varphi}}_{m_{\text{red}}} + \underbrace{k R^2 \dot{\varphi}}_{k_{\text{red}}} + \underbrace{\left(\frac{4R^2}{c_1} + \frac{4R^2}{c_2} \right) \cdot \varphi}_{\frac{1}{c_{\text{red}}}} = 0$$

• a redukált rezgőrendszer:



$$m_{\text{red}} \ddot{q} + k_{\text{red}} \dot{q} + \frac{1}{c_{\text{red}}} q = 0$$

b) ① $\frac{d}{dt} \left(\frac{\partial E}{\partial \dot{q}} \right)$ meghatározása

$$\begin{aligned} E &= \frac{1}{2} J_a \omega^2 = \frac{1}{2} \left(\frac{3}{2} m R^2 \right) \cdot \omega^2 = \\ &= \frac{1}{2} \left(\frac{3}{2} m R^2 \right) \cdot \left(\frac{\dot{y}_B}{2R} \right)^2 = \frac{1}{2} \left(\frac{3}{8} m \right) \dot{y}_B^2 \end{aligned}$$

$$\frac{\partial E}{\partial \dot{y}_B} = \frac{3}{8} m \dot{y}_B$$

$$\frac{d}{dt} \left(\frac{\partial E}{\partial \dot{y}_B} \right) = \frac{3}{8} m \ddot{y}_B$$

$$\frac{d}{dt} \left(\frac{\partial E}{\partial \dot{q}} \right) = \frac{3}{8} m \ddot{y}_B$$

④ Q_k meghatározása

$$Q_k = \vec{F}_k \cdot \vec{\beta}_s$$

$$\vec{F}_k = -k \cdot \vec{v}_{\text{ang}} = -k \cdot \vec{v}_s =$$

$$= -k \left(\frac{\dot{y}_B}{2} \right) \vec{j} = -k \left(\frac{\dot{y}_B}{2} \right) \vec{j}$$

$$\vec{\beta}_s = \frac{\partial \vec{v}_s}{\partial \dot{y}_B} = \frac{\partial \left(\frac{\dot{y}_B}{2} \right)}{\partial \dot{y}_B} = \frac{1}{2} \vec{j}$$

$$Q_k = -k \frac{\dot{y}_B}{2} \vec{j} \cdot \frac{1}{2} \vec{j} = -k \cdot \frac{1}{4} \cdot \dot{y}_B$$

$$Q_k = -k \cdot \frac{1}{4} \cdot \dot{y}_B$$

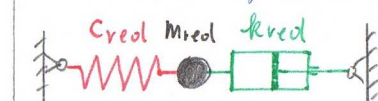
• lehelyettesítés Lagrange egyenletbe

$$\frac{3}{8} m \ddot{y}_B - 0 = - \left(\frac{1}{c_1} + \frac{1}{c_2} \right) y_B - \frac{k}{4} \cdot \dot{y}_B$$

• rendezés az egyenletet

$$\underbrace{\frac{3}{8} m \ddot{y}_B}_{m_{\text{red}}} + \underbrace{\frac{k}{4} \cdot \dot{y}_B}_{k_{\text{red}}} + \underbrace{\left(\frac{1}{c_1} + \frac{1}{c_2} \right) y_B}_{\frac{1}{c_{\text{red}}}} = 0$$

• a redukált rezgőrendszer:



$$m_{\text{red}} \ddot{q} + k_{\text{red}} \dot{q} + \frac{1}{c_{\text{red}}} q = 0$$

a) ① $\frac{d}{dt} \left(\frac{\partial E}{\partial \dot{q}} \right)$ meghatározása

$$E = \frac{1}{2} J_a \dot{\varphi}^2 = \frac{1}{2} \left(\frac{3}{2} m R^2 \right) \dot{\varphi}^2$$

$$\frac{\partial E}{\partial \dot{\varphi}} = \frac{3}{2} m R^2 \dot{\varphi}$$

$$\frac{d}{dt} \frac{3}{2} m R^2 \dot{\varphi} = \frac{3}{2} m R^2 \ddot{\varphi}$$

$$\frac{d}{dt} \left(\frac{\partial E}{\partial \dot{q}} \right) = \frac{3}{2} m R^2 \ddot{\varphi}$$

$$\text{② } \frac{\partial E}{\partial q} = 0$$

③ Q_c meghatározása

$$U = U_1 + U_2 = \frac{1}{2} \cdot \frac{y_B^2}{c_1} + \frac{1}{2} \cdot \frac{y_B^2}{c_2} =$$

$$= \frac{1}{2} \cdot \frac{(2R\varphi)^2}{c_1} + \frac{1}{2} \cdot \frac{(2R\varphi)^2}{c_2} =$$

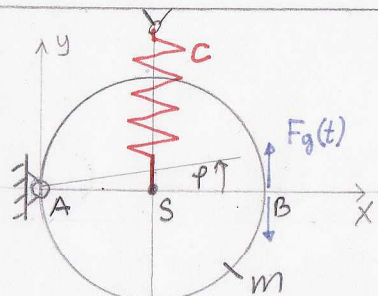
$$= \frac{1}{2} \cdot \left(\frac{4R^2 \varphi^2}{c_1} + \frac{4R^2 \varphi^2}{c_2} \right) = \frac{2R^2 \varphi^2}{c_1} + \frac{2R^2 \varphi^2}{c_2}$$

$$Q_c = - \frac{\partial U}{\partial \varphi} = - \frac{\partial}{\partial \varphi} \left(\frac{2R^2 \varphi^2}{c_1} + \frac{2R^2 \varphi^2}{c_2} \right) =$$

1.2

EGYSZABADSÁGFOKÚ REZGŐRENDSZER

MOZGÁSEGYENLETE



adott: $c, m, R, F_g(t) = F_{g0} \cdot \sin(\omega t + \varepsilon)$

• az általános koordináta:
 $q = r \quad \dot{q} = \dot{r}(\omega) \quad \ddot{q} = \ddot{r}(\varepsilon)$

• Lagrange egyenlet

$$\frac{d}{dt} \left[\frac{\partial E}{\partial \dot{q}} \right] - \frac{\partial E}{\partial q} = Q_c + Q_g$$

①. $\frac{d}{dt} \left[\frac{\partial E}{\partial \dot{q}} \right]$ meghatározása

$$E = \frac{1}{2} \mathcal{I}_A \omega^2 = \frac{1}{2} \left(\frac{3}{2} m R^2 \right) \dot{r}^2$$

$$\frac{d}{dt} \left[\frac{\partial E}{\partial \dot{q}} \right] = \frac{3}{2} m R^2 \ddot{r}$$

②. $\frac{\partial E}{\partial q} = 0$

③. Q_c meghatározása

$$U = \frac{1}{2} \cdot \frac{(Rr)^2}{c} = \frac{1}{2} \frac{R^2}{c} \cdot r^2$$

$$Q_c = - \frac{\partial U}{\partial r} = - \frac{R^2}{c} \cdot r$$

$$Q_c = - \frac{R^2}{c} \cdot r$$

④. Q_g meghatározása

$$Q_g = \vec{F}_g \cdot \vec{\beta}_B$$

$$\vec{F}_g = F_{g0} \cdot \sin(\omega t + \varepsilon) \vec{j}$$

$$\vec{\beta}_B = \frac{\partial \vec{v}_B}{\partial \dot{r}} = \frac{\partial (2R\dot{r} \vec{j})}{\partial \dot{r}} = 2R \vec{j}$$

$$Q_g = \vec{F}_g \cdot \vec{\beta}_B = 2R F_g(t)$$

$$Q_g = 2R F_g(t)$$

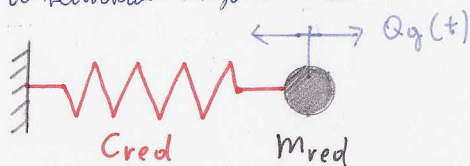
• leheletlen Lagrange egyenlet:

$$\frac{3}{2} m R^2 \ddot{r} - 0 = - \frac{R^2}{c} \cdot r + 2R F_g(t)$$

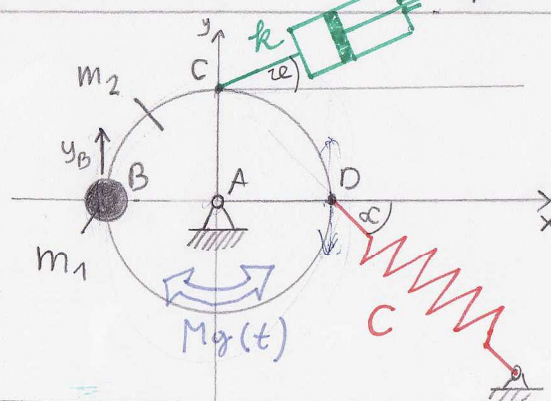
• rendezve az egyenletet:

$$\underbrace{\frac{3}{2} m R^2}_{m_{red}} \ddot{r} + \underbrace{\frac{R^2}{c}}_{\frac{1}{c_{red}}} \cdot r = \underbrace{2R F_g(t)}_{Q_g(t)}$$

• a redukált rezgőrendszer:



$$m_{red} \cdot \ddot{r} + \frac{1}{c_{red}} \cdot r = Q_g(t)$$



adott: $m_1, m_2, c, k, \ell, \varphi, R, M_g(t) = M_{g0} \cdot \sin(\omega t + \varepsilon)$

• az általános koordináta:

$$q = u_B \quad \dot{q} = \dot{u}_B(\varphi) \quad \ddot{q} = \ddot{u}_B(\alpha)$$

• Lagrange egyenlet

$$\frac{d}{dt} \left[\frac{\partial E}{\partial \dot{q}} \right] - 0 = Q_c + Q_k + Q_g$$

①. $\frac{d}{dt} \left[\frac{\partial E}{\partial \dot{q}} \right]$ meghatározása

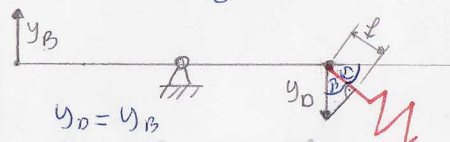
$$\begin{aligned} E &= E_1 + E_2 = \frac{1}{2} m_1 v_1^2 + \frac{1}{2} \mathcal{I}_A \omega^2 \\ &= \frac{1}{2} m_1 \dot{u}_B^2 + \frac{1}{2} \left(\frac{1}{2} m_2 R^2 \right) \cdot \left(\frac{\dot{u}_B}{R} \right)^2 \\ &= \frac{1}{2} \left(m_1 + \frac{1}{2} m_2 R^2 \frac{1}{R^2} \right) \cdot \dot{u}_B^2 \end{aligned}$$

$$\frac{d}{dt} \left[\frac{\partial E}{\partial \dot{q}} \right] = \left(m_1 + \frac{1}{2} m_2 \right) \ddot{u}_B$$

②. $\frac{\partial E}{\partial q} = 0$

③. Q_c meghatározása

$$U = \frac{1}{2} \cdot \frac{(u_B \cdot \cos \beta)^2}{c} = \frac{1}{2} \cdot \frac{\cos^2 \beta}{c} \cdot u_B^2$$



$$u_D = u_B$$

$$\beta = 90^\circ - \alpha$$

$$Q_c = - \frac{\partial U}{\partial u_B} = - \frac{\cos^2 \beta}{c} \cdot u_B$$

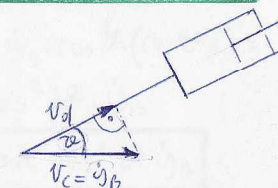
$$Q_c = - \frac{\cos^2 \beta}{c} \cdot u_B$$

④. Q_k meghatározása

$$\begin{aligned} D &= \frac{1}{2} k u_D^2 = \frac{1}{2} k (\dot{u}_B \cos \varphi)^2 = \\ &= \frac{1}{2} k \cdot \dot{u}_B^2 \cdot \cos^2 \varphi \end{aligned}$$

$$Q_k = - \frac{\partial D}{\partial \dot{u}_B} = - k \cdot \cos^2 \varphi \cdot \dot{u}_B$$

$$Q_k = - k \cdot \cos^2 \varphi \cdot \dot{u}_B$$



⑤. Q_g meghatározása

$$Q_g = \vec{M}_g(t) \cdot \vec{b}$$

$$\vec{M}_g(t) = M_g(t) \vec{k} = M_{g0} \sin(\omega t + \varepsilon) \vec{k}$$

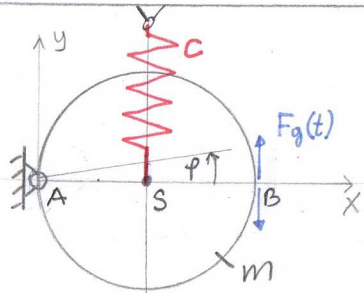
$$\vec{b} = \frac{\partial \vec{\omega}}{\partial \dot{u}_B} = \frac{\partial \left(\frac{\dot{u}_B}{R} \vec{k} \right)}{\partial \dot{u}_B} = \frac{1}{R} \vec{k}$$

$$Q_g = \frac{1}{R} M_g(t)$$

• leheletlen Lagrange egyenlet

1.2

EGYSZABADSÁGFOKÚ REZGŐRENDSZER



adott: $c, m, R, F_g(t) = F_{g0} \sin(\omega t + \varepsilon)$

• az általános koordináta:
 $q = \varphi \quad \dot{q} = \dot{\varphi}(\omega) \quad \ddot{q} = \ddot{\varphi}(\varepsilon)$

• Lagrange egyenlet

$$\frac{d}{dt} \left[\frac{\partial E}{\partial \dot{q}} \right] - \frac{\partial E}{\partial q} = Q_c + Q_g$$

1. $\frac{d}{dt} \left[\frac{\partial E}{\partial \dot{q}} \right]$ meghatározása

$$E = \frac{1}{2} \vartheta_a \omega^2 = \frac{1}{2} \left(\frac{3}{2} m R^2 \right) \dot{\varphi}^2$$

$$\frac{d}{dt} \left[\frac{\partial E}{\partial \dot{q}} \right] = \frac{3}{2} m R^2 \ddot{\varphi}$$

2. $\frac{\partial E}{\partial q} = 0$

3. Q_c meghatározása

$$U = \frac{1}{2} \cdot \frac{(R\varphi)^2}{c} = \frac{1}{2} \frac{R^2}{c} \cdot \varphi^2$$

$$Q_c = - \frac{\partial U}{\partial \varphi} = - \frac{R^2}{c} \cdot \varphi$$

$$Q_c = - \frac{R^2}{c} \cdot \varphi$$

4. Q_g meghatározása

$$Q_g = \vec{F}_g \cdot \vec{\beta}_B$$

$$\vec{F}_g = F_{g0} \cdot \sin(\omega t + \varepsilon) \vec{j}$$

$$\vec{\beta}_B = \frac{\partial \vec{r}_B}{\partial \dot{\varphi}} = \frac{\partial (2R\dot{\varphi} \vec{j})}{\partial \dot{\varphi}} = 2R\vec{j}$$

$$Q_g = \vec{F}_g \cdot \vec{\beta}_B = 2R F_g(t)$$

$$Q_g = 2R F_g(t)$$

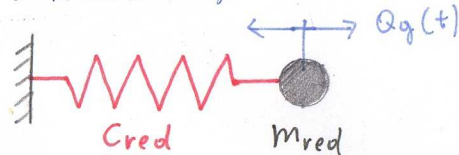
• behelyettesítve Lagrange egyenletbe:

$$\frac{3}{2} m R^2 \ddot{\varphi} - 0 = - \frac{R^2}{c} \cdot \varphi + 2R F_g(t)$$

• rendezve az egyenletet:

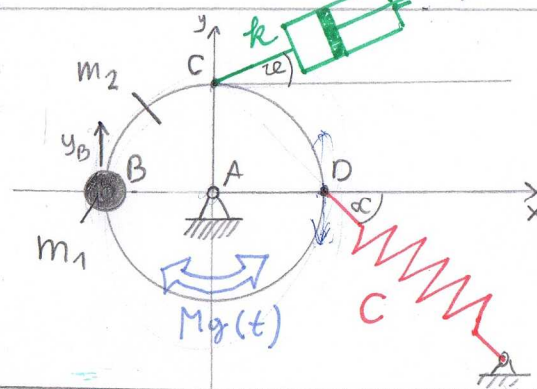
$$\underbrace{\frac{3}{2} m R^2 \ddot{\varphi}}_{m_{red}} + \underbrace{\frac{R^2}{c} \cdot \varphi}_{\frac{1}{c_{red}}} = \underbrace{2R F_g(t)}_{Q_g(t)}$$

• a redukált rezgőrendszer:



$$m_{red} \ddot{q} + \frac{1}{c_{red}} \cdot q = Q_g(t)$$

MOZGÁSEGYENLETE



adott: $m_1, m_2, c, k, l, \vartheta, R, M_g(t) = M_{g0} \sin(\omega t + \varepsilon)$

• az általános koordináta:

$$q = y_B \quad \dot{q} = \dot{y}_B \quad \ddot{q} = \ddot{y}_B$$

• Lagrange egyenlet

$$\frac{d}{dt} \left[\frac{\partial E}{\partial \dot{q}} \right] - 0 = Q_c + Q_k + Q_g$$

1. $\frac{d}{dt} \left[\frac{\partial E}{\partial \dot{q}} \right]$ meghatározása

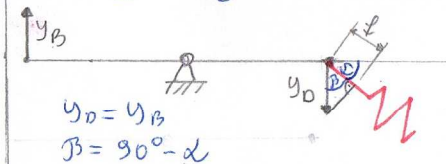
$$E = E_1 + E_2 = \frac{1}{2} m_1 v_1^2 + \frac{1}{2} \vartheta_a \omega^2 = \frac{1}{2} m_1 \dot{y}_B^2 + \frac{1}{2} \left(\frac{1}{2} m_2 R^2 \right) \cdot \left(\frac{\dot{y}_B}{R} \right)^2 = \frac{1}{2} \left(m_1 + \frac{1}{2} m_2 R^2 \frac{1}{R^2} \right) \cdot \dot{y}_B^2$$

$$\frac{d}{dt} \left[\frac{\partial E}{\partial \dot{q}} \right] = \left(m_1 + \frac{1}{2} m_2 \right) \ddot{y}_B$$

2. $\frac{\partial E}{\partial q} = 0$

3. Q_c meghatározása

$$U = \frac{1}{2} \cdot \frac{(y_B \cdot \cos \beta)^2}{c} = \frac{1}{2} \cdot \frac{\cos^2 \beta}{c} \cdot y_B^2$$



$$y_B = y_B$$

$$\beta = 90^\circ - \alpha$$

$$Q_c = - \frac{\partial U}{\partial y_B} = - \frac{\cos^2 \beta}{c} \cdot y_B$$

$$Q_c = - \frac{\cos^2 \beta}{c} \cdot y_B$$

4. Q_k meghatározása

$$Q_k = \vec{F}_k \cdot \vec{\beta}_C$$

$$\vec{F}_k = -k \cdot \vec{r}_{C/B}$$

$$|\vec{r}_{C/B}| = |\vec{r}_{B/C}| = |\dot{y}_B|$$

$$v_{C/B} = v_C \cdot \cos \vartheta = \dot{y}_B \cdot \cos \vartheta$$

$$\vec{r}_{C/B} = v_{C/B} \cdot (\cos \vartheta \vec{i} + \sin \vartheta \vec{j})$$

$$\vec{\beta}_C = \frac{\partial \vec{r}_C}{\partial \dot{y}_B} = \frac{\partial (\dot{y}_B \vec{i})}{\partial \dot{y}_B} = \vec{i}$$

$$Q_k = -k \cdot \dot{y}_B \cdot \cos \vartheta \cdot (\cos \vartheta \vec{i} + \sin \vartheta \vec{j}) \cdot \vec{i} = -k \cdot \cos^2 \vartheta \cdot \dot{y}_B$$

$$Q_k = -k \cos^2 \vartheta \cdot \dot{y}_B$$

5. Q_g meghatározása

$$Q_g = \vec{M}_g(t) \cdot \vec{b}$$

$$\vec{M}_g(t) = M_g(t) \vec{k} = M_{g0} \sin(\omega t + \varepsilon) \vec{k}$$

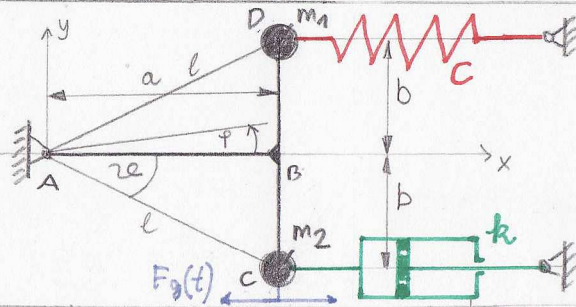
$$\vec{b} = \frac{\partial \vec{r}}{\partial \dot{y}_B} = \frac{\partial \left(\frac{y_B}{R} \vec{k} \right)}{\partial \dot{y}_B} = \frac{1}{R} \vec{k}$$

$$Q_g = \frac{1}{R} M_g(t)$$

• behelyettesítve Lagrange egyenletbe

1.3

EGY SZABADSAÁGFOKÚ REZGŐRENDSZER MOZGÁSEGYENLETE

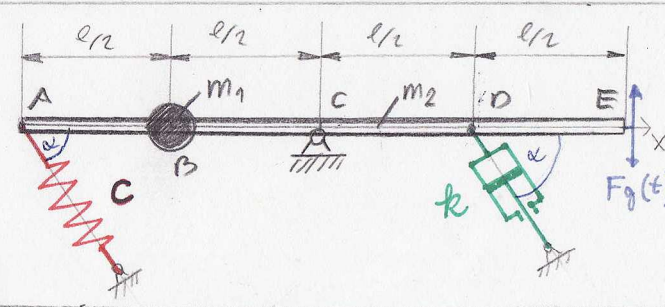


adott: $m_1, m_2, a, b, c, k,$
 $F_g(t) = F_{g0} \sin(\omega t + \epsilon)$

az általános koordináta:
 $q = \varphi \quad \dot{q} = \dot{\varphi} \quad \ddot{q} = \ddot{\varphi}$

Lagrange egyenlet:

$$\frac{d}{dt} \left[\frac{\partial E}{\partial \dot{q}} \right] - \frac{\partial E}{\partial q} = Q_c + Q_k + Q_g$$



adott: $m_1, m_2, l, c, k, d,$
 $F_g(t) = F_{g0} \sin(\omega t + \epsilon)$

általános koordináta:
 $q = \varphi \quad \dot{q} = \dot{\varphi} \quad \ddot{q} = \ddot{\varphi}$

Lagrange egyenlet

$$\frac{d}{dt} \left[\frac{\partial E}{\partial \dot{q}} \right] - \frac{\partial E}{\partial q} = Q_c + Q_k + Q_g$$

1. $\frac{d}{dt} \left[\frac{\partial E}{\partial \dot{q}} \right]$ meghatározása

$$E = \frac{1}{2} m_1 v_1^2 + \frac{1}{2} m_2 v_2^2 =$$

$$= \frac{1}{2} m_1 (\dot{\varphi}^2) + \frac{1}{2} m_2 (\dot{\varphi}^2) =$$

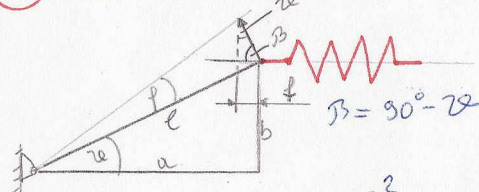
$$= \frac{1}{2} m_1 (a^2 + b^2) \dot{\varphi}^2 + \frac{1}{2} m_2 (a^2 + b^2) \dot{\varphi}^2 =$$

$$= \frac{1}{2} (m_1 + m_2) (a^2 + b^2) \dot{\varphi}^2$$

$$\frac{d}{dt} \left[\frac{\partial E}{\partial \dot{q}} \right] = (m_1 + m_2) (a^2 + b^2) \ddot{\varphi}$$

2. $\frac{\partial E}{\partial q} = 0$

3. Q_c meghatározása



$$U = \frac{1}{2} \frac{F^2}{c} = \frac{1}{2} \cdot \frac{(l \cdot \varphi \cdot \cos \beta)^2}{c} =$$

$$= \frac{1}{2} \cdot \frac{(a^2 + b^2) \cdot \varphi^2 \cdot \cos^2 \beta}{c} =$$

$$= \frac{1}{2} \cdot \frac{(a^2 + b^2) \cdot \cos^2 \beta}{c} \cdot \varphi^2$$

$$Q_c = - \frac{\partial U}{\partial \varphi} = - \frac{(a^2 + b^2) \cos^2 \beta}{c} \cdot \varphi$$

$$Q_c = - \frac{(a^2 + b^2) \cos^2 \beta}{c} \cdot \varphi$$

4. Q_k meghatározása

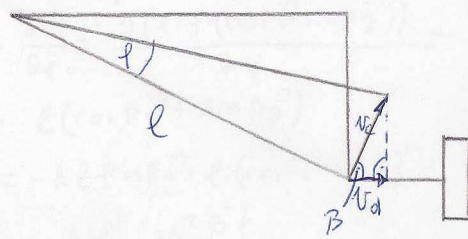
$$D = \frac{1}{2} k v_d^2 = \frac{1}{2} k \cdot (v_c \cos \beta)^2 =$$

$$= \frac{1}{2} k \cdot (\dot{\varphi} \cdot l \cdot \cos \beta)^2 =$$

$$= \frac{1}{2} k \cdot \varphi^2 \cdot l^2 \cdot \cos^2 \beta$$

$$Q_k = - \frac{\partial D}{\partial \varphi} = - k \cdot l^2 \cdot \cos^2 \beta \cdot \dot{\varphi}$$

$$Q_k = - k \cdot l^2 \cdot \cos^2 \beta \cdot \dot{\varphi}$$



5. Q_g meghatározása

$$Q_g = \vec{F}_g(t) \cdot \vec{p}_c$$

$$\vec{F}_g(t) = F_g(t) \vec{i} = F_{g0} \sin(\omega t + \epsilon) \vec{i}$$

$$\vec{p}_c = \frac{\partial \vec{r}_c}{\partial \varphi} = l (\cos \beta \vec{i} + \sin \beta \vec{j})$$

$$Q_g = F_g(t) \cdot l \cdot \cos \beta$$

1. $\frac{d}{dt} \left[\frac{\partial E}{\partial \dot{q}} \right]$ meghatározása

$$E = E_1 + E_2 = \frac{1}{2} m_1 v_1^2 + \frac{1}{2} m_2 v_2^2 =$$

$$= \frac{1}{2} m_1 \left(\frac{c}{2} \cdot \dot{\varphi} \right)^2 + \frac{1}{2} \left(\frac{1}{12} m_2 (2c)^2 \right) \dot{\varphi}^2 =$$

$$= \frac{1}{2} \cdot \left(\frac{m_1}{4} + \frac{m_2}{3} \right) \cdot c^2 \cdot \dot{\varphi}^2$$

$$\frac{d}{dt} \left[\frac{\partial E}{\partial \dot{q}} \right] = \left(\frac{m_1}{4} + \frac{m_2}{3} \right) c^2 \cdot \ddot{\varphi}$$

2. $\frac{\partial E}{\partial q} = 0$

3. Q_c meghatározása

$$U = \frac{1}{2} \frac{F^2}{c} = \frac{1}{2} \cdot \frac{(l \cdot \varphi \cdot \cos \beta)^2}{c} =$$

$$= \frac{1}{2} \cdot \frac{l^2 \cos^2 \beta}{c} \cdot \varphi^2$$

$$Q_c = - \frac{\partial U}{\partial \varphi} = - \frac{l^2 \cos^2 \beta}{c} \cdot \varphi$$

$$Q_c = - \frac{l^2 \cos^2 \beta}{c} \cdot \varphi$$

4. Q_k meghatározása

$$Q_k = F_k \cdot \vec{p}_D$$

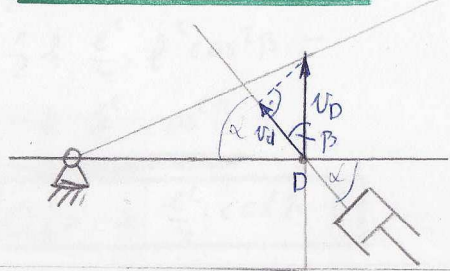
$$F_k = - k \cdot v_{D, \text{rel}}$$

$$D = \frac{1}{2} k \cdot v_d^2 = \frac{1}{2} k \cdot (v_b \cdot \cos \beta)^2 =$$

$$= \frac{1}{2} k \cdot \left(\frac{c}{2} \cdot \dot{\varphi} \cdot \cos \beta \right)^2 = \frac{1}{2} k \cdot \frac{c^2}{4} \cdot \dot{\varphi}^2 \cos^2 \beta$$

$$Q_k = - \frac{\partial D}{\partial \dot{\varphi}} = - k \cdot \frac{c^2}{4} \cdot \cos^2 \beta \cdot \dot{\varphi}$$

$$Q_k = - k \cdot \frac{c^2}{4} \cdot \cos^2 \beta \cdot \dot{\varphi}$$



5. Q_g meghatározása

$$Q_g = \vec{F}_g \cdot \vec{p}_E$$

$$\vec{F}_g = F_g(t) \vec{j} = F_{g0} \sin(\omega t + \epsilon) \vec{j}$$

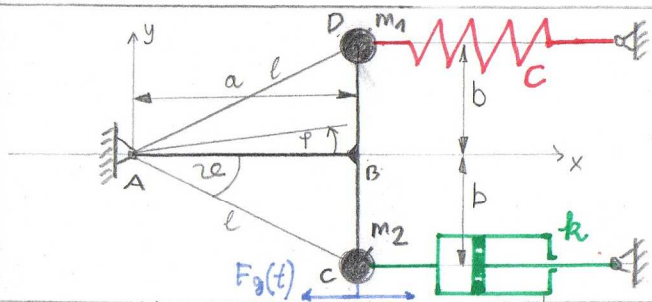
$$\vec{p}_E = \frac{\partial \vec{r}_E}{\partial \varphi} = \frac{\partial (l \cdot \varphi \cdot \vec{j})}{\partial \varphi} = l \cdot \vec{j}$$

$$Q_g = F_g(t) \vec{j} \cdot l \cdot \vec{j} = F_g(t) \cdot l$$

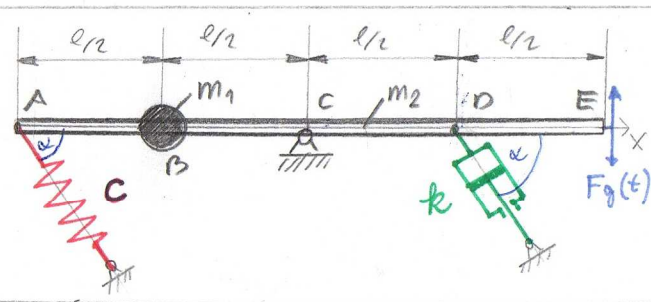
$$Q_g = F_g(t) \cdot l$$

1.3

EGYSZABA DSÁGFOKÚ REZGŐRENDSZER MOZGÁSEGYENLETE



adott: m_1, m_2, a, b, c, k, d
 $F_g(t) = F_{g0} \cdot \sin(\omega t + \varepsilon)$
 az általános koordináta:
 $q = \varphi, \dot{q} = \dot{\varphi}(\omega) \ddot{q} = \ddot{\varphi}(\varepsilon)$
 Lagrange egyenlet:
 $\frac{d}{dt} \left(\frac{\partial E}{\partial \dot{q}} \right) - \frac{\partial E}{\partial q} = Q_c + Q_k + Q_g$



adott: m_1, m_2, ℓ, c, k, d
 $F_g(t) = F_{g0} \sin(\omega t + \varepsilon)$
 az általános koordináta:
 $q = \varphi, \dot{q} = \dot{\varphi}(\omega) \ddot{q} = \ddot{\varphi}(\varepsilon)$
 Lagrange egyenlet:
 $\frac{d}{dt} \left(\frac{\partial E}{\partial \dot{q}} \right) - \frac{\partial E}{\partial q} = Q_c + Q_k + Q_g$

1. $\frac{d}{dt} \left(\frac{\partial E}{\partial \dot{q}} \right)$ meghatározása

$$E = \frac{1}{2} m_1 v_1^2 + \frac{1}{2} m_2 v_2^2 =$$

$$= \frac{1}{2} m_1 (\ell \dot{\varphi})^2 + \frac{1}{2} m_2 (\ell \dot{\varphi})^2 =$$

$$= \frac{1}{2} m_1 (a^2 + b^2) \dot{\varphi}^2 + \frac{1}{2} m_2 (a^2 + b^2) \dot{\varphi}^2 =$$

$$= \frac{1}{2} (m_1 + m_2) (a^2 + b^2) \dot{\varphi}^2$$

$$\frac{d}{dt} \left(\frac{\partial E}{\partial \dot{q}} \right) = (m_1 + m_2) (a^2 + b^2) \ddot{\varphi}$$

4. Q_k meghatározása

$$Q_k = \vec{F}_k \cdot \vec{\beta}_c$$

$$v_c = \ell \cdot \dot{\varphi}$$

$$\vec{v}_c = \ell \cdot \dot{\varphi} (\cos \beta \vec{i} + \sin \beta \vec{j})$$

$$v_{duy} = \ell \cdot \dot{\varphi} \cos \beta$$

$$\vec{v}_{duy} = \ell \cdot \dot{\varphi} \cos \beta \cdot \vec{i}$$

$$\vec{F}_k = -k \cdot \vec{v}_{duy} = -k \cdot \ell \cdot \dot{\varphi} \cos \beta \cdot \vec{i}$$

$$\vec{\beta}_c = \frac{\partial \vec{v}_c}{\partial \dot{\varphi}} = \frac{\partial (\ell \cdot \dot{\varphi} (\cos \beta \vec{i} + \sin \beta \vec{j}))}{\partial \dot{\varphi}} =$$

$$= \ell (\cos \beta \vec{i} + \sin \beta \vec{j})$$

$$Q_k = -k \ell \dot{\varphi} \cos \beta \cdot \ell (\cos \beta \vec{i} + \sin \beta \vec{j}) =$$

$$= -k \ell^2 \cos^2 \beta \dot{\varphi}$$

$$Q_k = -k \ell^2 \cos^2 \beta \dot{\varphi}$$

2. $\frac{\partial E}{\partial q} = 0$

3. Q_c meghatározása

$$U = \frac{1}{2} \frac{F^2}{c} = \frac{1}{2} \cdot \frac{(\ell \cdot \dot{\varphi} \cos \beta)^2}{c} =$$

$$= \frac{1}{2} \cdot \frac{(a^2 + b^2) \cdot \dot{\varphi}^2 \cos^2 \beta}{c} =$$

$$= \frac{1}{2} \cdot \frac{(a^2 + b^2) \cos^2 \beta}{c} \cdot \dot{\varphi}^2$$

$$Q_c = -\frac{\partial U}{\partial \varphi} = -\frac{(a^2 + b^2) \cos^2 \beta}{c} \cdot \dot{\varphi}$$

$$Q_c = -\frac{(a^2 + b^2) \cos^2 \beta}{c} \cdot \dot{\varphi}$$

5. Q_g meghatározása

$$Q_g = \vec{F}_g(t) \cdot \vec{\beta}_c$$

$$\vec{F}_g(t) = F_g(t) \vec{i} = F_{g0} \sin(\omega t + \varepsilon) \vec{i}$$

$$\vec{\beta}_c = \frac{\partial \vec{v}_c}{\partial \dot{\varphi}} = \ell (\cos \beta \vec{i} + \sin \beta \vec{j})$$

$$Q_g = F_g(t) \cdot \ell \cdot \cos \beta$$

$$Q_g = F_g(t) \cdot \ell \cdot \cos \beta$$

1. $\frac{d}{dt} \left(\frac{\partial E}{\partial \dot{q}} \right)$ meghatározása

$$E = E_1 + E_2 = \frac{1}{2} m_1 v_1^2 + \frac{1}{2} m_2 v_2^2 =$$

$$= \frac{1}{2} m_1 \left(\frac{\ell}{2} \dot{\varphi} \right)^2 + \frac{1}{2} \left(\frac{1}{12} m_2 (2\ell)^2 \right) \dot{\varphi}^2 =$$

$$= \frac{1}{2} \left(\frac{m_1}{4} + \frac{m_2}{3} \right) \ell^2 \dot{\varphi}^2$$

$$\frac{d}{dt} \left(\frac{\partial E}{\partial \dot{q}} \right) = \left(\frac{m_1}{4} + \frac{m_2}{3} \right) \ell^2 \ddot{\varphi}$$

2. $\frac{\partial E}{\partial q} = 0$

3. Q_c meghatározása

$$U = \frac{1}{2} \frac{F^2}{c} = \frac{1}{2} \cdot \frac{(\ell \cdot \dot{\varphi} \cos \beta)^2}{c} =$$

$$= \frac{1}{2} \cdot \frac{\ell^2 \cos^2 \beta}{c} \cdot \dot{\varphi}^2$$

$$Q_c = -\frac{\partial U}{\partial \varphi} = -\frac{\ell^2 \cos^2 \beta}{c} \cdot \dot{\varphi}$$

$$Q_c = -\frac{\ell^2 \cos^2 \beta}{c} \cdot \dot{\varphi}$$

4. Q_k meghatározása

$$Q_k = \vec{F}_k \cdot \vec{\beta}_c$$

$$\vec{F}_k = -k \cdot \vec{v}_{duy}$$

5. Q_g meghatározása

$$Q_g = \vec{F}_g \cdot \vec{\beta}_E$$

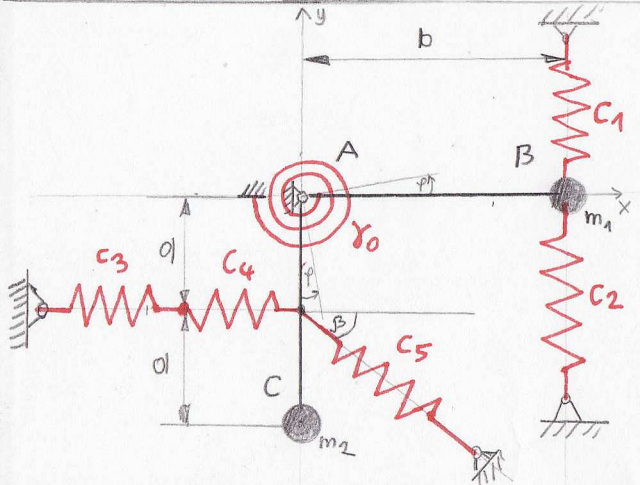
$$\vec{F}_g = F_g(t) \vec{j} = F_{g0} \sin(\omega t + \varepsilon) \vec{j}$$

$$\vec{\beta}_E = \frac{\partial \vec{v}_E}{\partial \dot{\varphi}} = \frac{\partial (\ell \cdot \dot{\varphi} \vec{j})}{\partial \dot{\varphi}} = \ell \cdot \vec{j}$$

$$Q_g = F_g(t) \vec{j} \cdot \ell \cdot \vec{j} = F_g(t) \cdot \ell$$

$$Q_g = F_g(t) \cdot \ell$$

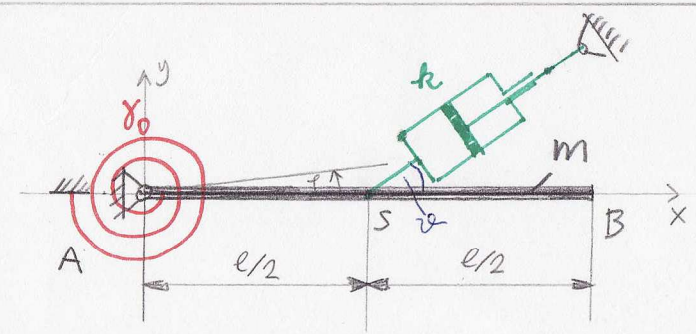
EGYSZABADSÁGFOKÚ REZGŐRENDSZER MOZGÁSEGYENLETE



adott: $m_1, m_2, a, b, c_1, c_2, c_3, c_4, c_5, \gamma_0, \beta$

- az általános koordináta:
 $q = \varphi$ (szögelfordulás)
 $\dot{q} = \dot{\varphi}(\omega)$ (szögsebesség)
 $\ddot{q} = \ddot{\varphi}(\varepsilon)$ (szöggyorsulás)

• Lagrange egyenlet:
$$\frac{d}{dt} \left[\frac{\partial E}{\partial \dot{q}} \right] - \frac{\partial E}{\partial q} = Q_c$$



adott: $m, e, k, \gamma_0, \varphi$
• az általános koordináta
 $q = \varphi$ $\dot{q} = \dot{\varphi}(\omega)$ $\ddot{q} = \ddot{\varphi}(\varepsilon)$

• Lagrange egyenlet:
$$\frac{d}{dt} \left[\frac{\partial E}{\partial \dot{q}} \right] - \frac{\partial E}{\partial q} = Q_c + Q_k$$

1. $\frac{d}{dt} \left[\frac{\partial E}{\partial \dot{q}} \right]$ meghatározása

$$E = \frac{1}{2} m_1 v_1^2 + \frac{1}{2} m_2 v_2^2 = \frac{1}{2} m_1 (b \dot{\varphi})^2 + \frac{1}{2} m_2 (2a \dot{\varphi})^2 = \frac{1}{2} (m_1 b^2 + m_2 4a^2) \dot{\varphi}^2$$

$$\frac{d}{dt} \left[\frac{\partial E}{\partial \dot{q}} \right] = (m_1 b^2 + m_2 4a^2) \ddot{\varphi}$$

$$Q_c = - \left(\frac{b^2}{c_1} + \frac{b^2}{c_2} + \frac{a^2}{c_3+c_4} + \frac{a^2 \cos^2 \beta}{c_5} + \frac{1}{\gamma_0} \right) \varphi$$

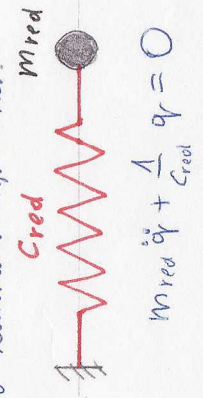
2. $\frac{\partial E}{\partial q} = 0$

3. Q_c meghatározása

$$U = U_1 + U_2 + U_3 + U_4 + U_5 + U_6 = \frac{1}{2} \cdot \frac{(b\varphi)^2}{c_1} + \frac{1}{2} \cdot \frac{(b\varphi)^2}{c_2} + \frac{1}{2} \cdot \frac{(a\varphi)^2}{c_3+c_4} + \frac{1}{2} \cdot \frac{(a\varphi \cos \beta)^2}{c_5} + \frac{1}{2} \cdot \frac{\varphi^2}{\gamma_0} = \frac{1}{2} \cdot \left[\frac{b^2}{c_1} + \frac{b^2}{c_2} + \frac{a^2}{c_3+c_4} + \frac{a^2 \cos^2 \beta}{c_5} + \frac{1}{\gamma_0} \right] \varphi^2$$

$$Q_c = - \frac{\partial U}{\partial \varphi}$$

• behelyettesítés a Lagrange egyenletbe
 $(m_1 b^2 + m_2 4a^2) \ddot{\varphi} - 0 = - \left(\frac{b^2}{c_1} + \frac{b^2}{c_2} + \frac{a^2}{c_3+c_4} + \frac{a^2 \cos^2 \beta}{c_5} + \frac{1}{\gamma_0} \right) \varphi$
• rendezés az egyenletet
 $(m_1 b^2 + m_2 4a^2) \ddot{\varphi} + \left(\frac{b^2}{c_1} + \frac{b^2}{c_2} + \frac{a^2}{c_3+c_4} + \frac{a^2 \cos^2 \beta}{c_5} + \frac{1}{\gamma_0} \right) \varphi = 0$



1. $\frac{d}{dt} \left[\frac{\partial E}{\partial \dot{q}} \right]$ meghatározása

$$E = \frac{1}{2} J_a \omega^2 = \frac{1}{2} \left(\frac{1}{3} m l^2 \right) \dot{\varphi}^2$$

$$\frac{d}{dt} \left[\frac{\partial E}{\partial \dot{q}} \right] = \frac{1}{3} m l^2 \ddot{\varphi}$$

2. $\frac{\partial E}{\partial q} = 0$

3. Q_c meghatározása

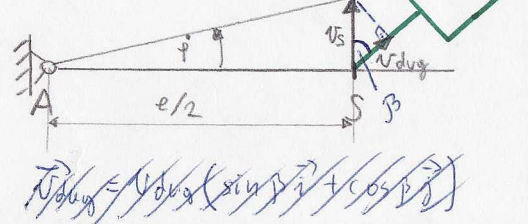
$$U = \frac{1}{2} \cdot \frac{\varphi^2}{\gamma_0} \quad Q_c = - \frac{\partial U}{\partial \varphi}$$

$$Q_c = - \frac{1}{\gamma_0} \cdot \varphi$$

4. Q_k meghatározása

$$Q_k = \vec{F}_k \cdot \vec{\beta}$$
$$\vec{F}_k = -k \cdot \vec{u}_{dk}$$

$$u_{dk} = \frac{l}{2} \cdot \dot{\varphi} \cdot \cos \beta$$



$$D = \frac{1}{2} k u_d^2 = \frac{1}{2} k (u_s \cdot \cos \beta)^2 = \frac{1}{2} k \cdot \left(\frac{l}{2} \dot{\varphi} \cos \beta \right)^2 = \frac{1}{2} k \cdot \frac{l^2}{4} \cdot \dot{\varphi}^2 \cos^2 \beta$$

$$Q_k = - \frac{\partial D}{\partial \varphi} = -k \cdot \frac{l^2}{4} \cdot \cos^2 \beta \cdot \dot{\varphi}$$

$$Q_k = -k \frac{l^2}{4} \cdot \cos^2 \beta \cdot \dot{\varphi}$$

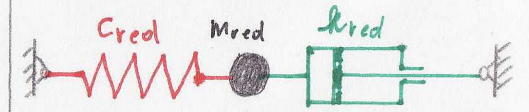
• behelyettesítés a Lagrange egyenletbe
 $\frac{1}{3} m l^2 \ddot{\varphi} - 0 = - \frac{1}{\gamma_0} \varphi - k \frac{l^2}{4} \cos^2 \beta \cdot \dot{\varphi}$

• rendezés az egyenletet

$$\frac{1}{3} m l^2 \ddot{\varphi} + k \frac{l^2}{4} \cos^2 \beta \cdot \dot{\varphi} + \frac{1}{\gamma_0} \varphi = 0$$

M_{red} k_{red} $\frac{1}{C_{red}}$

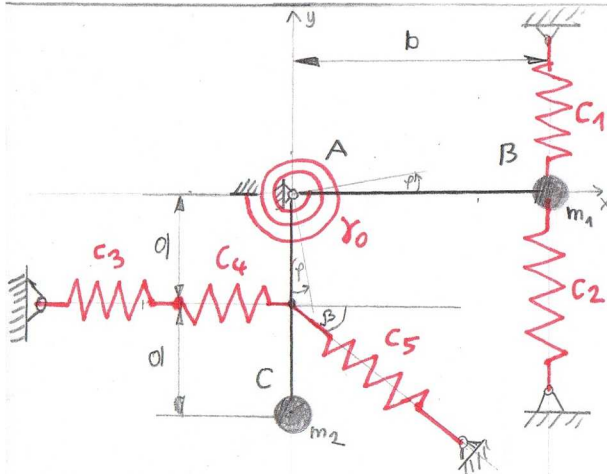
• a redukált rezgőrendszer



$$M_{red} \ddot{\varphi} + k_{red} \dot{\varphi} + \frac{1}{C_{red}} \varphi = 0$$

1.4

EGYSZABA DSÁGFOKÚ REZGŐRENDSZER MOZGÁSEGYENLETE



adott: $m_1, m_2, a, b, c_1, c_2, c_3, c_4, c_5, \gamma_0, \beta$

- az általános koordináta:
 $q = p$ (megtekintés)
 $\dot{q} = \dot{p}(\omega)$ (megtekintés)
 $\ddot{q} = \ddot{p}(\epsilon)$ (megtekintés)

• Lagrange egyenlet:

$$\frac{d}{dt} \left[\frac{\partial E}{\partial \dot{q}} \right] - \frac{\partial E}{\partial q} = Q_c$$

1. $\frac{d}{dt} \left[\frac{\partial E}{\partial \dot{q}} \right]$ meghatározása

$$E = \frac{1}{2} m_1 v_1^2 + \frac{1}{2} m_2 v_2^2 =$$

$$= \frac{1}{2} m_1 (b \dot{p})^2 + \frac{1}{2} m_2 (2a \dot{p})^2 =$$

$$= \frac{1}{2} (m_1 b^2 + m_2 4a^2) \dot{p}^2$$

$$\frac{d}{dt} \left[\frac{\partial E}{\partial \dot{q}} \right] = (m_1 b^2 + m_2 4a^2) \ddot{p}$$

2. $\frac{\partial E}{\partial q} = 0$

3. Q_c meghatározása

$$U = U_1 + U_2 + U_3 + U_4 + U_5 + U_6 =$$

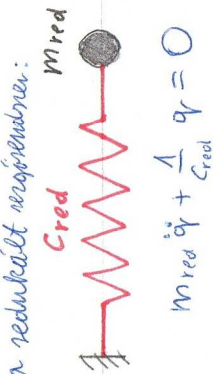
$$= \frac{1}{2} \cdot \frac{(b p)^2}{c_1} + \frac{1}{2} \cdot \frac{(b p)^2}{c_2} + \frac{1}{2} \cdot \frac{(a p)^2}{c_3 + c_4} +$$

$$+ \frac{1}{2} \cdot \frac{(a p \cos \beta)^2}{c_5} + \frac{1}{2} \cdot \frac{p^2}{\gamma_0} =$$

$$= \frac{1}{2} \cdot \left[\frac{b^2}{c_1} + \frac{b^2}{c_2} + \frac{a^2}{c_3 + c_4} + \frac{a^2 \cos^2 \beta}{c_5} + \frac{1}{\gamma_0} \right] p^2$$

$$Q_c = - \frac{\partial U}{\partial p}$$

$$Q_c = - \left(\frac{b^2}{c_1} + \frac{b^2}{c_2} + \frac{a^2}{c_3 + c_4} + \frac{a^2 \cos^2 \beta}{c_5} + \frac{1}{\gamma_0} \right) p$$



1. $\frac{d}{dt} \left[\frac{\partial E}{\partial \dot{q}} \right]$ meghatározása

$$E = \frac{1}{2} J_a \omega^2 = \frac{1}{2} \left(\frac{1}{3} m e^2 \right) \dot{p}^2$$

$$\frac{d}{dt} \left[\frac{\partial E}{\partial \dot{q}} \right] = \frac{1}{3} m e^2 \ddot{p}$$

2. $\frac{\partial E}{\partial q} = 0$

3. Q_c meghatározása

$$U = \frac{1}{2} \cdot \frac{p^2}{\gamma_0} \quad Q_c = - \frac{\partial U}{\partial p}$$

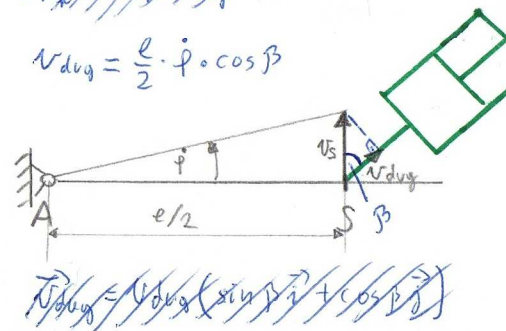
$$Q_c = - \frac{1}{\gamma_0} \cdot p$$

4. Q_k meghatározása

$$Q_k = \vec{F}_k \cdot \vec{p}$$

$$\vec{F}_k = -k \cdot \vec{u}_{du}$$

$$u_{du} = \frac{e}{2} \cdot \dot{p} \cdot \cos \beta$$



adott: m, e, k, γ_0, β

az általános koordináta
 $q = p \quad \dot{q} = \dot{p}(\omega) \quad \ddot{q} = \ddot{p}(\epsilon)$

• Lagrange egyenlet:

$$\frac{d}{dt} \left[\frac{\partial E}{\partial \dot{q}} \right] - \frac{\partial E}{\partial q} = Q_c + Q_k$$

$$\vec{F}_k = -k \cdot \vec{u}_{du} = -k \cdot \frac{e}{2} \cdot \dot{p} \cdot \cos \beta \cdot$$

$$\cdot (\sin \beta \vec{i} + \cos \beta \vec{j}) =$$

$$= -k \cdot \frac{e}{2} \cdot \dot{p} (\cos \beta \sin \beta \vec{i} + \cos^2 \beta \vec{j}) =$$

$$\vec{p} = \frac{\partial u_s}{\partial \dot{p}} = \frac{\partial \left(\frac{e}{2} \cdot \dot{p} \right)}{\partial \dot{p}} = \frac{e}{2} \vec{j}$$

$$Q_k = -k \cdot \frac{e}{2} \cdot \dot{p} (\cos \beta \sin \beta \vec{i} + \cos^2 \beta \vec{j}) \cdot \frac{e}{2} \vec{j} =$$

$$= -k \cdot \frac{e}{2} \cdot \dot{p} \cdot \cos^2 \beta \cdot \frac{e}{2} = -k \cdot \frac{e^2}{4} \cdot \cos^2 \beta \cdot \dot{p}$$

$$Q_k = -k \frac{e^2}{4} \cdot \cos^2 \beta \cdot \dot{p}$$

• behelyettesítés a Lagrange egyenletbe

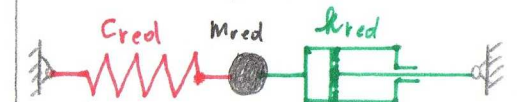
$$\frac{1}{3} m e^2 \ddot{p} - 0 = - \frac{1}{\gamma_0} \cdot p - k \frac{e^2}{4} \cdot \cos^2 \beta \cdot \dot{p}$$

• rendezés az egyenletet

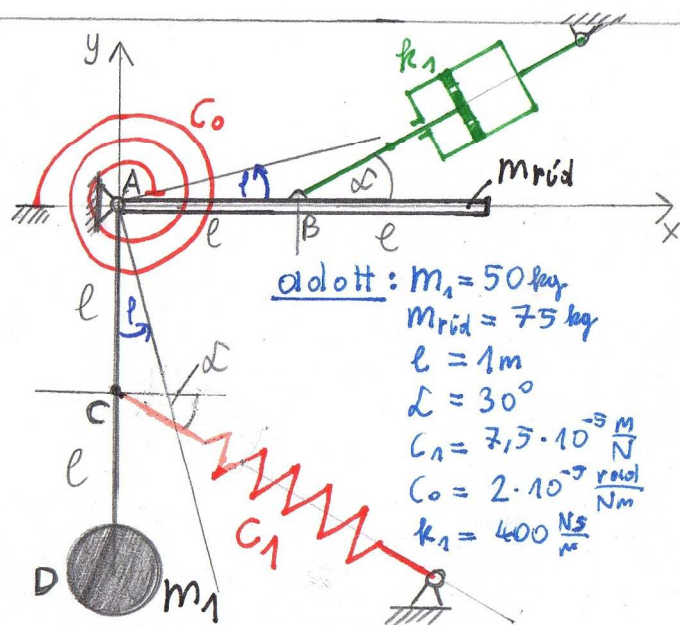
$$\frac{1}{3} m e^2 \ddot{p} + k \frac{e^2}{4} \cdot \cos^2 \beta \cdot \dot{p} + \frac{1}{\gamma_0} \cdot p = 0$$

$$m_{red} \quad k_{red} \quad \frac{1}{c_{red}}$$

• a redukált rezgőrendszer



$$m_{red} \ddot{p} + k_{red} \dot{p} + \frac{1}{c_{red}} p = 0$$



• általános koordináta:

$q = \varphi$ - szögelfordulás

$\dot{q} = \dot{\varphi} = \omega$ - szögsebesség

$\ddot{q} = \ddot{\varphi} = \varepsilon$ - szöggyorsulás

• Lagrange egyenlet:

$$\frac{d}{dt} \left[\frac{\partial E}{\partial \dot{q}} \right] - \frac{\partial E}{\partial q} = Q_c + Q_k$$

• Vissza helyettesítve a Lagrange egyenletbe:

$$m_{\text{red}} \ddot{\varphi} - 0 = -\frac{1}{c_{\text{red}}} \varphi - k_{\text{red}} \varphi$$

$$m_{\text{red}} \ddot{\varphi} + k_{\text{red}} \varphi + \frac{1}{c_{\text{red}}} \varphi = 0$$

$$300 \ddot{\varphi} + 100 \varphi + 60000 \varphi = 0$$

$\frac{d}{dt} \left[\frac{\partial E}{\partial \dot{q}} \right]$ meghatározása

$$E = \frac{1}{2} m_1 v_D^2 + \frac{1}{2} J_A \omega^2$$

$$E = \frac{1}{2} m_1 \dot{x}_D^2 + \frac{1}{2} \left[\frac{1}{3} m_{\text{rid}} (2l)^2 \right] \dot{\varphi}^2$$

$$E = \frac{1}{2} m_1 (2l \dot{\varphi})^2 + \frac{1}{2} \left[\frac{1}{3} m_{\text{rid}} 4l^2 \right] \dot{\varphi}^2$$

$$E = \frac{1}{2} \left[m_1 4l^2 + \frac{4}{3} m_{\text{rid}} l^2 \right] \dot{\varphi}^2$$

$$\frac{\partial E}{\partial \dot{q}} = \frac{\partial}{\partial \dot{\varphi}} \left[\frac{1}{2} \left[m_1 4l^2 + \frac{4}{3} m_{\text{rid}} l^2 \right] \dot{\varphi}^2 \right]$$

$$= \left[m_1 4l^2 + \frac{4}{3} m_{\text{rid}} l^2 \right] \dot{\varphi}$$

$$\frac{d}{dt} \left[\frac{\partial E}{\partial \dot{q}} \right] = \underbrace{\left[m_1 4l^2 + \frac{4}{3} m_{\text{rid}} l^2 \right]}_{m_{\text{red}}} \ddot{\varphi}$$

$$m_{\text{red}} = m_1 4l^2 + \frac{4}{3} m_{\text{rid}} l^2 =$$

$$= 50 \cdot 4 \cdot 1^2 + \frac{4}{3} 75 \cdot 1^2 = 300 \text{ kg m}^2$$

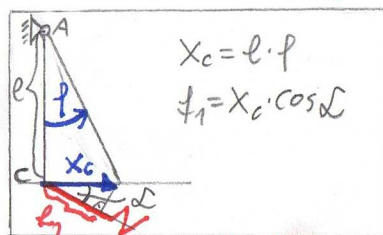
$$\frac{d}{dt} \left[\frac{\partial E}{\partial \dot{q}} \right] - \frac{\partial E}{\partial q} = Q_c + Q_k$$

Q_c meghatározása

$$U = U_0 + U_1$$

$$U = \frac{1}{2} \frac{p^2}{c_0} + \frac{1}{2} \frac{f_1^2}{c_1}$$

$$U = \frac{1}{2} \frac{p^2}{c_0} + \frac{1}{2} \frac{(l \varphi \cos \alpha)^2}{c_1}$$



$$U = \frac{1}{2} \left[\frac{1}{c_0} + \frac{l^2 \cos^2 \alpha}{c_1} \right] p^2$$

$$Q_c = -\frac{\partial U}{\partial \varphi} = -\frac{\partial U}{\partial p} = -\left[\frac{1}{c_0} + \frac{l^2 \cos^2 \alpha}{c_1} \right] p$$

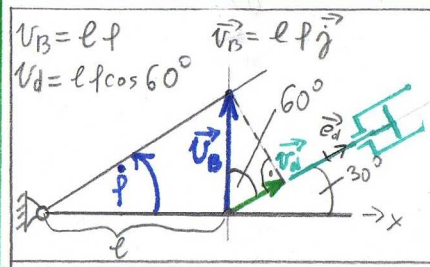
$$Q_c = -\underbrace{\left[\frac{1}{c_0} + \frac{l^2 \cos^2 \alpha}{c_1} \right]}_{\frac{1}{c_{\text{red}}}} p$$

$$\frac{1}{c_{\text{red}}} = \frac{1}{c_0} + \frac{l^2 \cos^2 \alpha}{c_1} =$$

$$= \frac{1}{2 \cdot 10^5} + \frac{1^2 \cos^2 30^\circ}{7,5 \cdot 10^5} = 6 \cdot 10^{-5} \text{ Nm}$$

$$c_{\text{red}} = \frac{1}{6 \cdot 10^{-5}} = 1,6 \cdot 10^5 \frac{\text{N}}{\text{m}}$$

Q_k meghatározása



$$\vec{v}_D = v_D \vec{e}_D = l \dot{\varphi} \cos 60^\circ (\cos 30^\circ \vec{i} + \sin 30^\circ \vec{j})$$

$$\vec{v}_D = l \dot{\varphi} \frac{1}{2} \left(\frac{\sqrt{3}}{2} \vec{i} + \frac{1}{2} \vec{j} \right)$$

$$\vec{v}_D = \frac{l \dot{\varphi}}{4} \sqrt{3} \vec{i} + \frac{l \dot{\varphi}}{4} \vec{j}$$

$$\vec{F}_k = -k \vec{v}_D = -k \left(\frac{\sqrt{3}}{4} l \vec{i} + \frac{1}{4} l \vec{j} \right) \dot{\varphi}$$

$$\vec{p}_D = \frac{\partial \vec{v}_D}{\partial \dot{\varphi}} = \frac{\partial}{\partial \dot{\varphi}} \left(\frac{l \dot{\varphi}}{4} \sqrt{3} \vec{i} + \frac{l \dot{\varphi}}{4} \vec{j} \right) = \frac{l}{4} \sqrt{3} \vec{i} + \frac{l}{4} \vec{j}$$

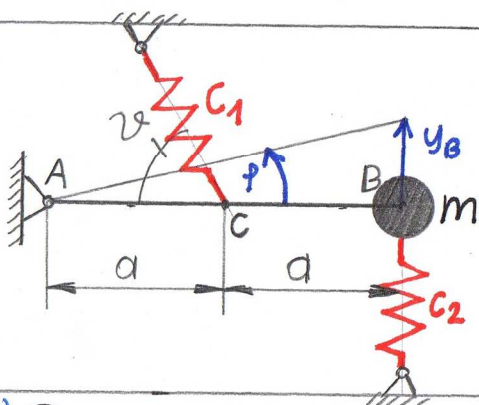
$$Q_k = \vec{F}_k \cdot \vec{p}_D = -k \left(\frac{\sqrt{3}}{4} l \vec{i} + \frac{1}{4} l \vec{j} \right) \dot{\varphi} \cdot \left(\frac{l}{4} \sqrt{3} \vec{i} + \frac{l}{4} \vec{j} \right)$$

$$= -k \frac{1}{4} l^2 \dot{\varphi}$$

$$Q_k = -\underbrace{\frac{1}{4} k l^2}_{k_{\text{red}}} \dot{\varphi}$$

$$k_{\text{red}} = -\frac{1}{4} k l^2 = -\frac{1}{4} \cdot 400 \cdot 1^2 =$$

$$= 100 \text{ Nms}$$



a) ① m_{red} meghatározása

$$E = \frac{1}{2} m v_B^2 = \frac{1}{2} m \dot{y}_B^2 = \frac{1}{2} m (2a \dot{\phi})^2 = \frac{1}{2} m 4a^2 \dot{\phi}^2$$

$$\frac{\partial E}{\partial \dot{\phi}} = \frac{\partial E}{\partial \dot{\phi}} = 2 \cdot \frac{1}{2} \cdot 4ma^2 \dot{\phi} = 4ma^2 \dot{\phi}$$

$$\frac{d}{dt} \left(\frac{\partial E}{\partial \dot{\phi}} \right) = \frac{d}{dt} 4ma^2 \dot{\phi} = 4ma^2 \ddot{\phi}$$

$$\boxed{\frac{d}{dt} \left(\frac{\partial E}{\partial \dot{\phi}} \right) = 4ma^2 \ddot{\phi} = m_{red} \ddot{\phi}}$$

② $\frac{\partial E}{\partial \phi} = \frac{\partial E}{\partial \phi} = 0$

③ C_{red} meghatározása

$$U = U_1 + U_2 = \frac{1}{2} \frac{f_1^2}{C_1} + \frac{1}{2} \frac{f_2^2}{C_2}$$

• adott: a, ℓ, m, C_1, C_2

• általános koordináta:

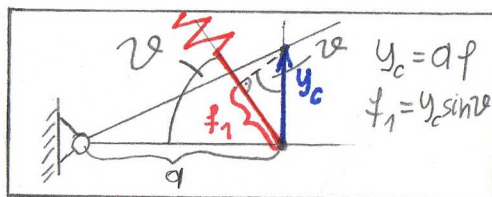
a) $q = \phi$
 $\dot{q} = \dot{\phi} = \omega$
 $\ddot{q} = \ddot{\phi} = \epsilon$

b) $q = y_B$
 $\dot{q} = \dot{y}_B = v_B$
 $\ddot{q} = \ddot{y}_B = a_B$

• Lagrange egyenlet:

$$\frac{d}{dt} \left(\frac{\partial E}{\partial \dot{q}} \right) - \frac{\partial E}{\partial q} = Q_c$$

$$= \frac{1}{2} \frac{(a \ell \sin \varphi)^2}{C_1} + \frac{1}{2} \frac{(2a \ell)^2}{C_2} =$$



$$= \frac{1}{2} \left[\frac{a^2 \sin^2 \varphi}{C_1} + \frac{4a^2}{C_2} \right] \phi^2$$

$$Q_c = -\frac{\partial U}{\partial \phi} = -\frac{\partial U}{\partial \phi} \frac{1}{2} \left(\frac{a^2 \sin^2 \varphi}{C_1} + \frac{4a^2}{C_2} \right) \phi^2 = -\left(\frac{a^2 \sin^2 \varphi}{C_1} + \frac{4a^2}{C_2} \right) \phi$$

$$\boxed{Q_c = -\left(\frac{a^2 \sin^2 \varphi}{C_1} + \frac{4a^2}{C_2} \right) \phi = -\frac{1}{C_{red}} \phi}$$

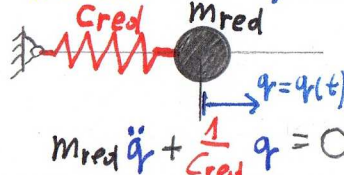
• behelyettesítve a Lagrange egyenletbe:

$$4ma^2 \ddot{\phi} - 0 = -\left(\frac{a^2 \sin^2 \varphi}{C_1} + \frac{4a^2}{C_2} \right) \phi$$

• átrendezve az egyenletet:

$$\underbrace{4ma^2}_{m_{red}} \ddot{\phi} + \underbrace{\left(\frac{a^2 \sin^2 \varphi}{C_1} + \frac{4a^2}{C_2} \right)}_{\frac{1}{C_{red}}} \phi = 0$$

• a redukált rezgőrendszer:



$$m_{red} \ddot{q} + \frac{1}{C_{red}} q = 0$$

b) ① m_{red} meghatározása

$$E = \frac{1}{2} m v_B^2 = \frac{1}{2} m \dot{y}_B^2$$

$$\frac{\partial E}{\partial \dot{q}} = \frac{\partial E}{\partial \dot{y}_B} = 2 \cdot \frac{1}{2} m \dot{y}_B = m \dot{y}_B$$

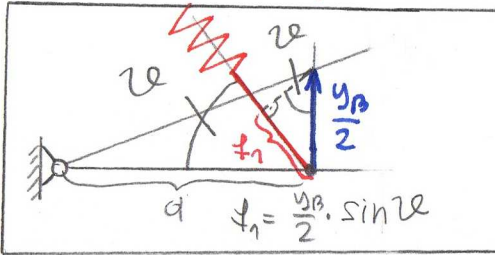
$$\frac{d}{dt} \left(\frac{\partial E}{\partial \dot{q}} \right) = \frac{d}{dt} m \dot{y}_B = m \ddot{y}_B$$

$$\boxed{\frac{d}{dt} \left(\frac{\partial E}{\partial \dot{q}} \right) = m \ddot{y}_B = m_{red} \ddot{y}_B}$$

② $\frac{\partial E}{\partial q} = \frac{\partial E}{\partial y_B} = 0$

③ C_{red} meghatározása

$$U = U_1 + U_2 = \frac{1}{2} \frac{f_1^2}{C_1} + \frac{1}{2} \frac{f_2^2}{C_2} =$$



$$= \frac{1}{2} \frac{\left(\frac{y_B}{2} \sin \varphi \right)^2}{C_1} + \frac{1}{2} \frac{y_B^2}{C_2} = \frac{1}{2} \left[\frac{\sin^2 \varphi}{4C_1} + \frac{1}{C_2} \right] y_B^2$$

$$Q_c = -\frac{\partial U}{\partial q} = -\frac{\partial U}{\partial y_B} \frac{1}{2} \left[\frac{\sin^2 \varphi}{4C_1} + \frac{1}{C_2} \right] y_B^2 = -\left[\frac{\sin^2 \varphi}{4C_1} + \frac{1}{C_2} \right] y_B$$

$$\boxed{Q_c = -\left[\frac{\sin^2 \varphi}{4C_1} + \frac{1}{C_2} \right] y_B = -\frac{1}{C_{red}} y_B}$$

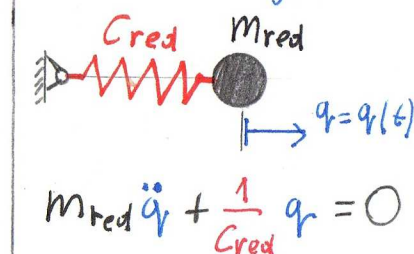
• behelyettesítve a Lagrange egyenletbe:

$$m \ddot{y}_B - 0 = -\left[\frac{\sin^2 \varphi}{4C_1} + \frac{1}{C_2} \right] y_B$$

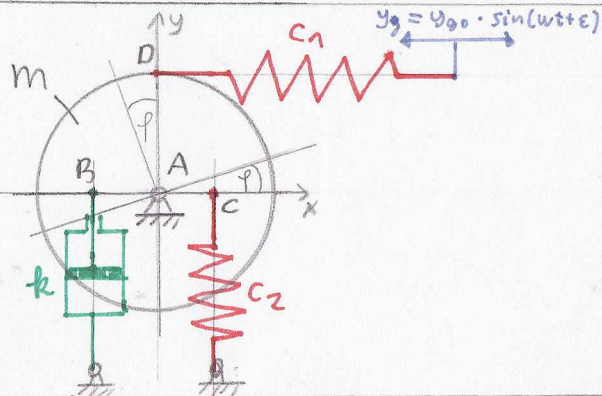
• átrendezve az egyenletet:

$$\underbrace{m}_{m_{red}} \ddot{y}_B + \underbrace{\left[\frac{\sin^2 \varphi}{4C_1} + \frac{1}{C_2} \right]}_{\frac{1}{C_{red}}} y_B = 0$$

• a redukált rezgőrendszer:



$$m_{red} \ddot{q} + \frac{1}{C_{red}} q = 0$$



adott: $m, R, k, c_1, c_2, y_g(t)$

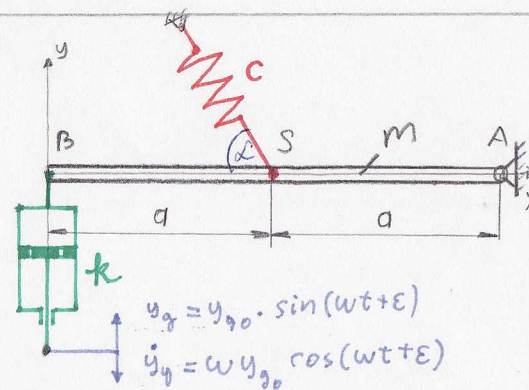
$$y_g(t) = y_{g0} \cdot \sin(\omega t + \epsilon)$$

az általános koordináta:

$$q = \varphi \quad \dot{q} = \dot{\varphi}(\omega) \quad \ddot{q} = \ddot{\varphi}(\epsilon)$$

Lagrange egyenlet

$$\frac{d}{dt} \left(\frac{\partial E}{\partial \dot{q}} \right) - \frac{\partial E}{\partial q} = Q_c + Q_k + Q_g$$



adott: $m, a, R, k, c, y_g(t), \alpha$

az általános koordináta

$$q = \varphi \quad \dot{q} = \dot{\varphi}(\omega) \quad \ddot{q} = \ddot{\varphi}(\epsilon)$$

Lagrange egyenlet:

$$\frac{d}{dt} \left(\frac{\partial E}{\partial \dot{q}} \right) - \frac{\partial E}{\partial q} = Q_c + Q_k + Q_g$$

1. $\frac{d}{dt} \left(\frac{\partial E}{\partial \dot{q}} \right)$ meghatározása

$$E = \frac{1}{2} \cdot J_a \cdot \omega^2 = \frac{1}{2} \cdot \left(\frac{1}{2} m R^2 \right) \cdot \dot{\varphi}^2$$

$$\frac{d}{dt} \left(\frac{\partial E}{\partial \dot{q}} \right) = \frac{1}{2} m R^2 \ddot{\varphi}$$

2. $\frac{\partial E}{\partial q} = 0$

3. 4. Q_c és Q_g meghatározása

$$U = \frac{1}{2} \frac{f_1^2}{c_1} + \frac{1}{2} \frac{f_2^2}{c_2} =$$

$$= \frac{1}{2} \frac{(R\varphi - y_g)^2}{c_1} + \frac{1}{2} \cdot \frac{\left(\frac{R}{2} \varphi \right)^2}{c_2}$$

$$Q_c + Q_g = - \frac{\partial U}{\partial \varphi} = - \frac{R\varphi - y_g}{c_1} \cdot R -$$

$$- \frac{R^2}{4c_2} \varphi = - \frac{R^2 \varphi}{c_1} + \frac{R y_g}{c_1} - \frac{R^2}{4c_2} \varphi$$

$$= - \left(\frac{R^2}{c_1} + \frac{R^2}{4c_2} \right) \varphi + \frac{R}{c_1} y_g(t)$$

$$Q_c = - \left(\frac{R^2}{c_1} + \frac{R^2}{4c_2} \right) \varphi$$

$$Q_g = \frac{R}{c_1} y_g(t)$$

5. Q_k meghatározása

$$D = \frac{1}{2} k v_d^2 = \frac{1}{2} k \cdot v_B^2 =$$

$$= \frac{1}{2} k \cdot \left(\frac{R}{2} \cdot \dot{\varphi} \right)^2 = \frac{1}{2} k \cdot \frac{R^2}{4} \cdot \dot{\varphi}^2$$

$$Q_k = - \frac{\partial D}{\partial \dot{\varphi}} = - k \frac{R^2}{4} \dot{\varphi}$$

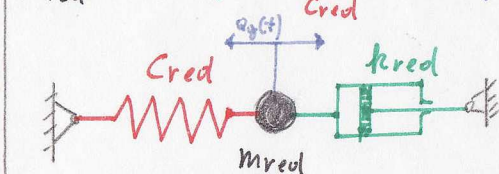
$$Q_k = - k \frac{R^2}{4} \dot{\varphi}$$

• behatározott Lagrange egyenlet

$$\frac{1}{2} m R^2 \ddot{\varphi} - 0 = - \left(\frac{R^2}{c_1} + \frac{R^2}{4c_2} \right) \varphi - k \frac{R^2}{4} \dot{\varphi} + \frac{R}{c_1} y_g(t)$$

• rendezve az egyenletet:

$$\frac{1}{2} m R^2 \ddot{\varphi} + \underbrace{k \frac{R^2}{4} \dot{\varphi}}_{k_{red}} + \underbrace{\left(\frac{R^2}{c_1} + \frac{R^2}{4c_2} \right) \varphi}_{\frac{1}{c_{red}}} = \underbrace{\frac{R}{c_1} y_g(t)}_{Q_g(t)}$$



$$m_{red} \ddot{q} + k_{red} \dot{q} + \frac{1}{c_{red}} q = Q_g(t)$$

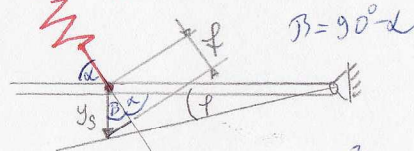
1. $\frac{d}{dt} \left(\frac{\partial E}{\partial \dot{q}} \right)$ meghatározása

$$E = \frac{1}{2} J_a \omega^2 = \frac{1}{2} \cdot \left(\frac{1}{3} m (2a)^2 \right) \dot{\varphi}^2$$

$$\frac{d}{dt} \left(\frac{\partial E}{\partial \dot{q}} \right) = \frac{4}{3} m a^2 \ddot{\varphi}$$

2. $\frac{\partial E}{\partial q} = 0$

3. Q_c meghatározása



$$U = \frac{1}{2} \cdot \frac{f^2}{c} = \frac{1}{2} \cdot \frac{(a\varphi \cos \beta)^2}{c} =$$

$$= \frac{1}{2} \cdot \frac{a^2 \cos^2 \beta}{c} \varphi^2$$

$$Q_c = - \frac{\partial U}{\partial \varphi} = - \frac{a^2 \cos^2 \beta}{c} \varphi$$

$$Q_c = - \frac{a^2 \cos^2 \beta}{c} \varphi$$

4. 5. Q_k és Q_g meghatározása

$$D = \frac{1}{2} k (v_B - v_g)^2 =$$

$$= \frac{1}{2} k (2a\dot{\varphi} - \dot{y}_g)^2$$

$$Q_k + Q_g = - \frac{\partial D}{\partial \dot{\varphi}} = - k (2a\dot{\varphi} - \dot{y}_g) \cdot 2a$$

$$= \underbrace{- k 4a^2 \dot{\varphi}}_{Q_k} + \underbrace{2a k \dot{y}_g}_{Q_g}$$

$$Q_k = - 4a^2 k \dot{\varphi}$$

$$Q_g = 2a k \dot{y}_g(t)$$

• behatározott Lagrange egyenlet:

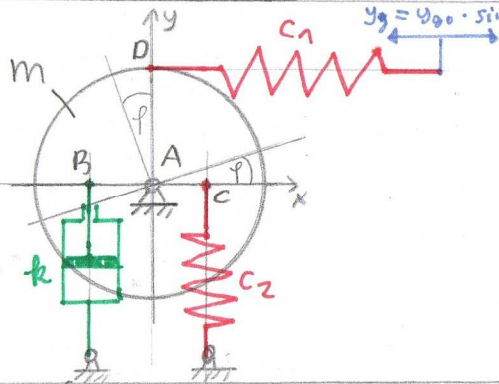
$$\frac{4}{3} m a^2 \ddot{\varphi} - 0 = - \frac{a^2 \cos^2 \beta}{c} \varphi - 4a^2 k \dot{\varphi} + 2a k \dot{y}_g(t)$$

rendezve az egyenletet:

$$\frac{4}{3} m a^2 \ddot{\varphi} + \underbrace{4a^2 k \dot{\varphi}}_{k_{red}} + \underbrace{\frac{a^2 \cos^2 \beta}{c} \varphi}_{\frac{1}{c_{red}}} = \underbrace{2a k \dot{y}_g(t)}_{Q_g(t)}$$

1.5

ÚTGERJESZTÉS



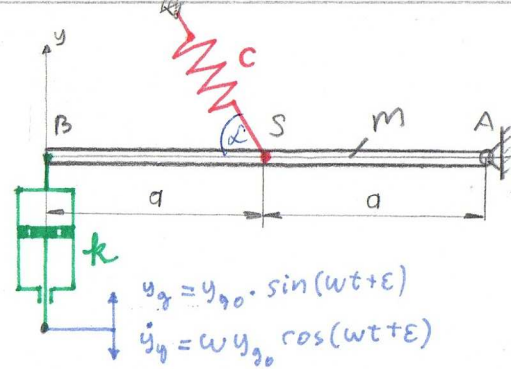
adott: $m, R, k, c_1, c_2, y_g(t)$

$$y_g(t) = y_{g0} \cdot \sin(\omega t + \varepsilon)$$

az általános koordináta:
 $q = \varphi \quad \dot{q} = \dot{\varphi}(\omega) \quad \ddot{q} = \ddot{\varphi}(\varepsilon)$

Lagrange egyenlet

$$\frac{d}{dt} \left(\frac{\partial E}{\partial \dot{q}} \right) - \frac{\partial E}{\partial q} = Q_c + Q_k + Q_g$$



adott: $m, a, R, k, C, y_g(t), \alpha$

az általános koordináta

$$q = \varphi \quad \dot{q} = \dot{\varphi}(\omega) \quad \ddot{q} = \ddot{\varphi}(\varepsilon)$$

Lagrange egyenlet:

$$\frac{d}{dt} \left(\frac{\partial E}{\partial \dot{q}} \right) - \frac{\partial E}{\partial q} = Q_c + Q_k + Q_g$$

1. $\frac{d}{dt} \left(\frac{\partial E}{\partial \dot{q}} \right)$ meghatározása

$$E = \frac{1}{2} \cdot J_a \cdot \omega^2 = \frac{1}{2} \cdot \left(\frac{1}{2} m R^2 \right) \cdot \dot{\varphi}^2$$

$$\frac{d}{dt} \left(\frac{\partial E}{\partial \dot{q}} \right) = \frac{1}{2} m R^2 \ddot{\varphi}$$

2. $\frac{\partial E}{\partial q} = 0$

3. 4. Q_c és Q_g meghatározása

$$U = \frac{1}{2} \frac{f_1^2}{c_1} + \frac{1}{2} \frac{f_2^2}{c_2} = \frac{1}{2} \frac{(R\varphi - y_g)^2}{c_1} + \frac{1}{2} \cdot \frac{\left(\frac{R}{2} \varphi \right)^2}{c_2}$$

$$Q_c + Q_g = - \frac{\partial U}{\partial \varphi} = - \frac{R\varphi - y_g}{c_1} \cdot R - \frac{R^2}{4c_2} \varphi = - \frac{R^2}{c_1} \varphi + \frac{R y_g}{c_1} - \frac{R^2}{4c_2} \varphi$$

$$= - \left(\frac{R^2}{c_1} + \frac{R^2}{4c_2} \right) \varphi + \frac{R}{c_1} y_g(t)$$

$$Q_c = - \left(\frac{R^2}{c_1} + \frac{R^2}{4c_2} \right) \varphi$$

$$Q_g = \frac{R}{c_1} y_g(t)$$

5. Q_k meghatározása

$$Q_k = \vec{F}_k \cdot \vec{\delta}_B$$

$$\vec{F}_k = -k \cdot \vec{u}_{B/A} = -k \cdot \left(-\frac{R}{2} \cdot \dot{\varphi} \right) \vec{j}$$

$$\vec{\delta}_B = \frac{\partial \vec{r}_B}{\partial \dot{\varphi}} = \frac{\partial \left(-\frac{R}{2} \cdot \dot{\varphi} \right)}{\partial \dot{\varphi}} = -\frac{R}{2} \vec{j}$$

$$Q_k = -k \left(-\frac{R}{2} \cdot \dot{\varphi} \right) \left(-\frac{R}{2} \right) = -k \frac{R^2}{4} \dot{\varphi}$$

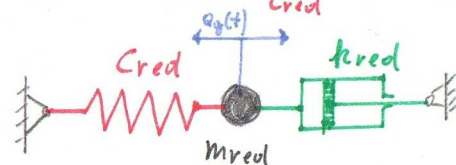
$$Q_k = -k \frac{R^2}{4} \dot{\varphi}$$

• behelyettesítés Lagrange egyenletbe

$$\frac{1}{2} m R^2 \ddot{\varphi} - 0 = - \left(\frac{R^2}{c_1} + \frac{R^2}{4c_2} \right) \varphi - k \frac{R^2}{4} \dot{\varphi} + \frac{R}{c_1} y_g(t)$$

• rendezés az egyenletet:

$$\frac{1}{2} m R^2 \ddot{\varphi} + k \frac{R^2}{4} \dot{\varphi} + \left(\frac{R^2}{c_1} + \frac{R^2}{4c_2} \right) \varphi = \frac{R}{c_1} y_g(t)$$



$$m_{red} \ddot{\varphi} + k_r \dot{\varphi} + \frac{1}{c_{red}} \varphi = Q_g(t)$$

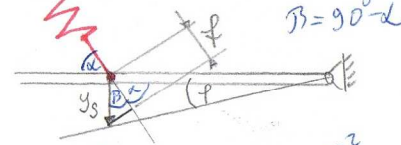
1. $\frac{d}{dt} \left(\frac{\partial E}{\partial \dot{q}} \right)$ meghatározása

$$E = \frac{1}{2} J_a \omega^2 = \frac{1}{2} \cdot \left(\frac{1}{3} m (2a)^2 \right) \dot{\varphi}^2$$

$$\frac{d}{dt} \left(\frac{\partial E}{\partial \dot{q}} \right) = \frac{4}{3} m a^2 \ddot{\varphi}$$

2. $\frac{\partial E}{\partial q} = 0$

3. Q_c meghatározása



$$U = \frac{1}{2} \cdot \frac{f^2}{c} = \frac{1}{2} \cdot \frac{(a\varphi \cos \beta)^2}{c} = \frac{1}{2} \cdot \frac{a^2 \cos^2 \beta}{c} \varphi^2$$

$$Q_c = - \frac{\partial U}{\partial \varphi} = - \frac{a^2 \cos^2 \beta}{c} \varphi$$

$$Q_c = - \frac{a^2 \cos^2 \beta}{c} \varphi$$

4. 5. Q_k és Q_g meghatározása

$$Q_k + Q_g = \vec{F}_k \cdot \vec{\delta}_B$$

$$\vec{F}_k = -k \cdot \vec{u}_{B/A} = -k (\dot{\varphi} - \dot{y}_g) \vec{j} = -k (2a\dot{\varphi} - \dot{y}_g) \vec{j}$$

$$\vec{\delta}_B = \frac{\partial \vec{r}_B}{\partial \dot{\varphi}} = \frac{\partial (2a\dot{\varphi})}{\partial \dot{\varphi}} = 2a \vec{j}$$

$$Q_k + Q_g = (-k 2a \dot{\varphi} + k \dot{y}_g) \vec{j} \cdot 2a \vec{j} = -4a^2 k \dot{\varphi} + 2a k \dot{y}_g(t)$$

$$Q_k = -4a^2 k \dot{\varphi}$$

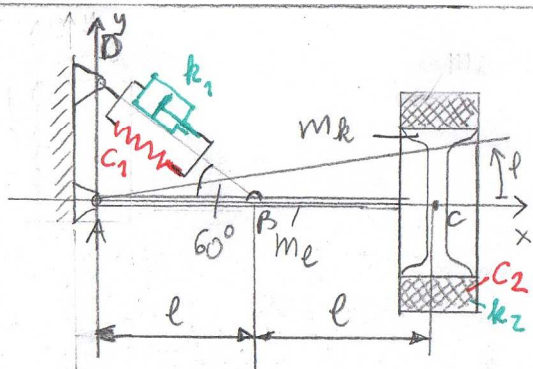
$$Q_g = 2a k \dot{y}_g(t)$$

• behelyettesítés Lagrange egyenletbe:

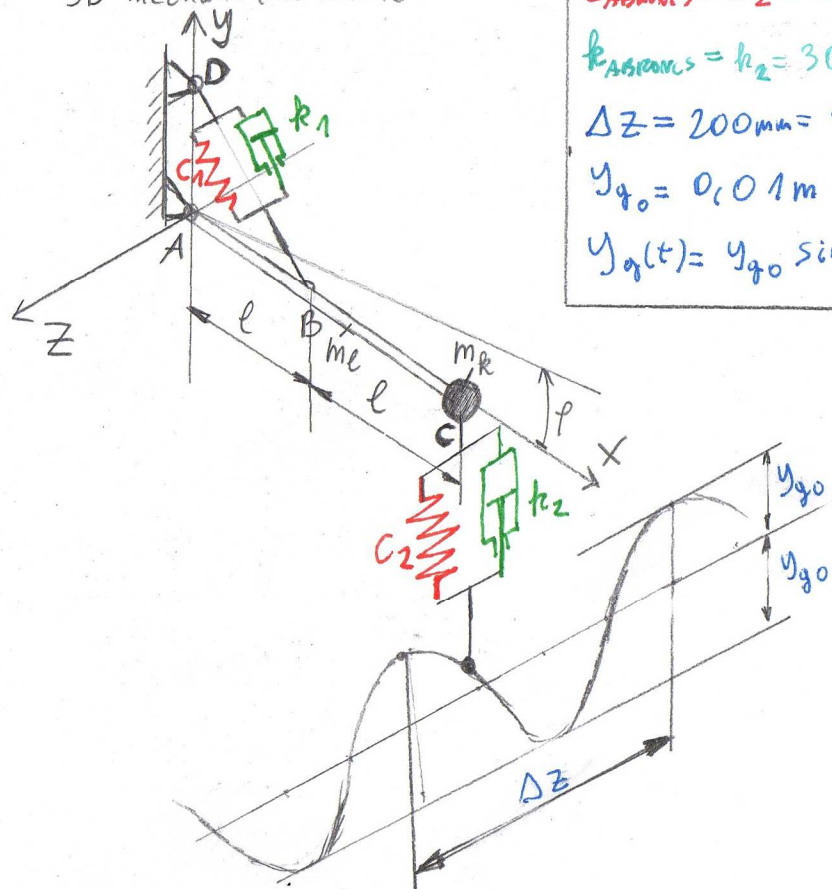
$$\frac{4}{3} m a^2 \ddot{\varphi} - 0 = - \frac{a^2 \cos^2 \beta}{c} \varphi - 4a^2 k \dot{\varphi} + 2a k \dot{y}_g$$

rendezés az egyenletet:

$$\frac{4}{3} m a^2 \ddot{\varphi} + 4a^2 k \dot{\varphi} + \frac{a^2 \cos^2 \beta}{c} \varphi = 2a k \dot{y}_g(t)$$



3D mechanikai modell



adott:

$$v_{\text{Autó}} = 36 \frac{\text{km}}{\text{h}} = 10 \frac{\text{m}}{\text{s}}$$

$$M_{\text{LENGŐKAR}} = M_c = 9 \text{ kg}$$

$$M_{\text{KERÉK}} = M_k = 20 \text{ kg}$$

$$c_1 = 4 \cdot 10^{-4} \frac{\text{m}}{\text{N}}$$

$$k_1 = 200 \frac{\text{N}}{\text{m}}$$

$$c_{\text{ABRÓCS}} = c_2 = 10^{-4} \frac{\text{m}}{\text{N}}$$

$$k_{\text{ABRÓCS}} = k_2 = 30 \frac{\text{N}}{\text{m}}$$

$$\Delta z = 200 \text{ mm} = 0,2 \text{ m}$$

$$y_{g0} = 0,01 \text{ m}$$

$$y_g(t) = y_{g0} \sin(\omega_g t)$$

• általános koordináta: $q = l$; $\dot{q} = \dot{l} = w$; $\ddot{q} = \ddot{l} = \varepsilon$

• gerjesztő függvény felírása:

- Ha a jármű $v_{\text{Autó}}$ sebességgel halad, akkor $T = \frac{\Delta z}{v_{\text{Autó}}}$ idő alatt teszi meg a két szomszédos emelkedés közötti utat, tehát a gerjesztés periódusideje:

$$T = \frac{\Delta z}{v_{\text{Autó}}} = \frac{0,2}{10} = 0,02 \text{ s}$$

- ebből a gerjesztés frekvenciája:

$$f = \frac{1}{T} = \frac{1}{0,02} = 50 \text{ Hz}$$

- innen a gerjesztés körfrekvenciája:

$$\omega_g = 2\pi f = 2\pi 50 = 314,159 \frac{\text{rad}}{\text{s}}$$

- így a gerjesztés elmozdulás függvénye:

$$y_g(t) = y_{g0} \sin(\omega_g t) = 0,01 \sin(314,159 t)$$

• A Lagrange-féle másodfajú mozgásegyenlet:

$$\frac{d}{dt} \left(\frac{\partial E}{\partial \dot{q}} \right) - \underbrace{\frac{\partial E}{\partial q}}_{=0} = Q_c + Q_k + Q_g$$

$$\text{ügerjesztés esetén: } Q_g = Q_{gc} + Q_{gk}$$

gerjesztés a rugó n keresztül

gerjesztés a csillapító n keresztül

$$\frac{d}{dt} \left(\frac{\partial E}{\partial \dot{f}} \right) = \underbrace{Q_c + Q_{yc}}_{\text{red}} + \underbrace{Q_k + Q_{yk}}_{\text{green}}$$

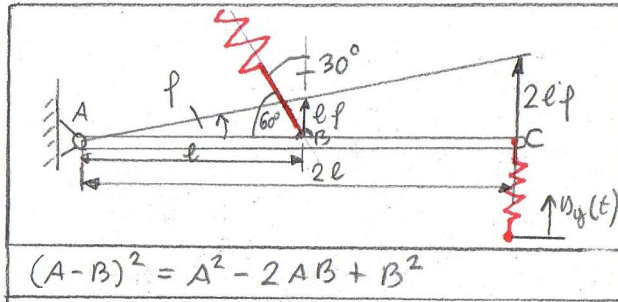
$\frac{d}{dt} \left(\frac{\partial E}{\partial \dot{f}} \right)$ meghatározása

$$\begin{aligned} E &= \frac{1}{2} J_a \omega^2 + \frac{1}{2} m_k v_c^2 \quad \rightarrow v_c = \dot{y}_c \\ &= \frac{1}{2} \left[\frac{1}{3} m_e (2l)^2 \right] \dot{f}^2 + \frac{1}{2} m_k (2l\dot{f})^2 = \\ &= \frac{1}{2} \left[\frac{1}{3} m_e 4l^2 + m_k 4l^2 \right] \dot{f} = \\ &\quad \text{Mred} \\ &= \frac{1}{2} M_{red} \dot{f} \end{aligned}$$

$$\frac{d}{dt} \left(\frac{\partial E}{\partial \dot{f}} \right) = M_{red} \ddot{f}$$

$Q_c + Q_{yc}$ meghatározása

$$\begin{aligned} U &= U_1 + U_2 = \frac{1}{2} \frac{f_1^2}{C_1} + \frac{1}{2} \frac{f_2^2}{C_2} = \\ &= \frac{1}{2} \frac{(l f \cos 30^\circ)^2}{C_1} + \frac{1}{2} \frac{(2l f - y_g(t))^2}{C_2} = \end{aligned}$$



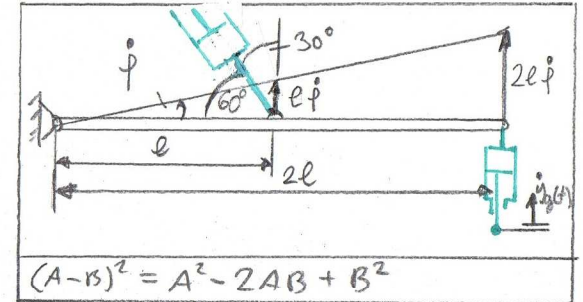
$$(A-B)^2 = A^2 - 2AB + B^2$$

$$= \frac{1}{2} \frac{l^2 f^2 \cos^2 30^\circ}{C_1} + \frac{1}{2} \frac{4l^2 f^2 - 4l f y_g(t) + y_g(t)^2}{C_2} =$$

$$\begin{aligned} Q_c + Q_{yc} &= - \frac{\partial U}{\partial f} = - \frac{1}{2} \cdot \frac{2l^2 f \cos^2 30^\circ}{C_1} - \\ &\quad - \frac{1}{2} \frac{8l^2 f - 4l y_g(t) + 0}{C_2} = \\ &= - \underbrace{\left[\frac{l^2 \cos^2 30^\circ}{C_1} + \frac{4l^2}{C_2} \right] f}_{\frac{1}{C_{red}}} + \underbrace{\frac{2l}{C_2} y_g(t)}_{Q_{yc}} \end{aligned}$$

$Q_k + Q_{yk}$ meghatározása

$$\begin{aligned} D &= D_1 + D_2 = \frac{1}{2} k_1 v_{d1}^2 + \frac{1}{2} k_2 v_{d2}^2 = \\ &= \frac{1}{2} k_1 (l \dot{f} \cos 30^\circ)^2 + \frac{1}{2} k_2 (2l \dot{f} - \dot{y}_g(t))^2 = \end{aligned}$$



$$= \frac{1}{2} k_1 (l^2 \dot{f}^2 \cos^2 30^\circ) + \frac{1}{2} k_2 (4l^2 \dot{f}^2 - 4l \dot{f} \dot{y}_g(t) + \dot{y}_g(t)^2)$$

$$\begin{aligned} Q_k + Q_{yk} &= - \frac{\partial D}{\partial \dot{f}} = - \frac{1}{2} k_1 (2l^2 \dot{f} \cos^2 30^\circ) - \\ &\quad - \frac{1}{2} k_2 (8l^2 \dot{f} - 4l \dot{y}_g(t) + 0) = \\ &= - \underbrace{\left[k_1 l^2 \cos^2 30^\circ + k_2 4l^2 \right] \dot{f}}_{k_{red}} + \underbrace{k_2 2l \dot{y}_g(t)}_{Q_{yk}} \end{aligned}$$

• Vissza helyettesítős a Lagrange egyenletbe:

$$\frac{d}{dt} \left(\frac{\partial E}{\partial \dot{f}} \right) = \underbrace{Q_c}_{\text{red}} + \underbrace{Q_{yc}}_{\text{red}} + \underbrace{Q_k}_{\text{red}} + \underbrace{Q_{yk}}_{\text{red}}$$

$$\left(\frac{1}{3} m_c 4l^2 + m_k 4l^2 \right) \ddot{f} = - \left(\frac{l^2 \cos^2 30^\circ}{c_1} + \frac{4l^2}{c_2} \right) f + \frac{2l}{c_2} y_g(t) - \left[k_1 l^2 \cos^2 30^\circ + k_2 4l^2 \right] f + k_2 2l \dot{y}_g(t)$$

• átrendezve:

$$\underbrace{\left(\frac{1}{3} m_c 4l^2 + m_k 4l^2 \right)}_{m_{red}} \ddot{f} + \underbrace{\left[k_1 l^2 \cos^2 30^\circ + 4k_2 l^2 \right]}_{k_{red}} f + \underbrace{\left(\frac{l^2 \cos^2 30^\circ}{c_1} + \frac{4l^2}{c_2} \right)}_{\frac{1}{c_{red}}} f = \underbrace{2k_2 l \dot{y}_g(t)}_{Q_{yk}} + \underbrace{\frac{2l}{c_2} y_g(t)}_{Q_{yc}}$$

• a reduktált rendszer jellemzői:

$$m_{red} = \frac{1}{3} m_c 4l^2 + m_k 4l^2 = \frac{1}{3} \cdot 9 \cdot 4 \cdot 0,5^2 + 20 \cdot 4 \cdot 0,5^2 = 23 \text{ kgm}^2$$

$$k_{red} = k_1 l^2 \cos^2 30^\circ + 4k_2 l^2 = 200 \cdot 0,5^2 \cos^2 30^\circ + 4 \cdot 30 \cdot 0,5^2 = 67,5 \text{ Nsm}$$

$$\frac{1}{c_{red}} = \frac{l^2 \cos^2 30^\circ}{c_1} + \frac{4l^2}{c_2} = \frac{0,5^2 \cos^2 30^\circ}{4 \cdot 10^{-4}} + \frac{4 \cdot 0,5^2}{10^{-4}} = 10468,75 \text{ N.m}$$

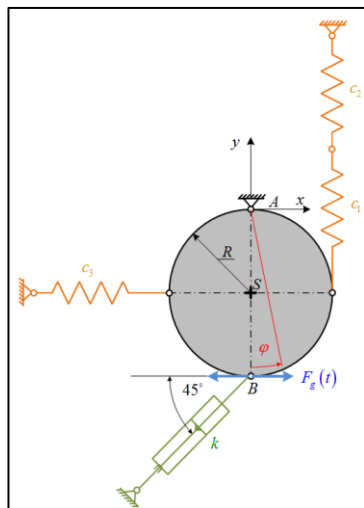
$$c_{red} = 10468,75^{-1} = 9,55 \cdot 10^{-5} \frac{\text{rad}}{\text{N.m}}$$

• behelyettesítve a Lagrange egyenletbe:

$$23 \ddot{f} + 67,5 \dot{f} + 10468,75 f = 94,2 \cos(314,159 t) + 100 \sin(314,159 t)$$

• az általános gerjesztőerő:

$$\begin{aligned} Q_y &= 2k_2 l \frac{d}{dt} y_{g0} \sin(\omega_y t) + \frac{2l}{c_2} y_{g0} \sin(\omega_y t) = \\ &= 2k_2 l \omega_y y_{g0} \cos(\omega_y t) + \frac{2l}{c_2} y_{g0} \sin(\omega_y t) = \\ &= 2l y_{g0} \left(k_2 \omega_y \cos(\omega_y t) + \frac{1}{c_2} \sin(\omega_y t) \right) = \\ &= 2 \cdot 0,5 \cdot 0,01 \left(30 \cdot 314,159 \cdot \cos(314,159 t) + \right. \\ &\quad \left. + \frac{1}{10^{-4}} \sin(314,159 t) \right) = 94,2 \cos(314,159 t) + \\ &\quad + 100 \sin(314,159 t) \end{aligned}$$



adott :

$$R = 0,5m$$

$$m = 20kg$$

$$c1 = 2 \cdot 10^{-5} m / N$$

$$c2 = 4 \cdot 10^{-5} m / N$$

$$c3 = 5 \cdot 10^{-5} m / N$$

$$k = 500 Ns / m$$

$$F_g(t) = F_{g0} \sin \omega t$$

$$F_{g0} = 200 N$$

$$\omega = 10 rad / s$$

$$q = \varphi$$

általános koordináta :

$$q = \varphi$$

$$\dot{q} = \dot{\varphi} = \omega$$

$$\ddot{q} = \ddot{\varphi} = \varepsilon$$

$$\frac{d}{dt} \left(\frac{\partial E}{\partial \dot{q}} \right)$$

+

$$\frac{\partial E}{\partial q}$$

=

$$Q_c$$

+

$$Q_k$$

+

$$Q_g$$

$\frac{d}{dt} \left(\frac{\partial E}{\partial \dot{q}} \right)$ meghatározása

$$E = \frac{1}{2} J_s \omega^2 = \frac{1}{2} J_s \dot{\varphi}^2 = \frac{1}{2} \left(\frac{3}{2} m R^2 \right) \dot{\varphi}^2$$

$$\frac{\partial E}{\partial \dot{q}} = \frac{\partial}{\partial \dot{\varphi}} \left(\frac{3}{2} m R^2 \dot{\varphi}^2 \right) = 3 m R^2 \dot{\varphi}$$

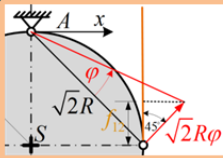
$$\frac{d}{dt} \left(\frac{\partial E}{\partial \dot{q}} \right) = \frac{d}{dt} 3 m R^2 \dot{\varphi} = 3 m R^2 \ddot{\varphi}$$

$$\frac{d}{dt} \left(\frac{\partial E}{\partial \dot{q}} \right) = \frac{3}{2} m R^2 \ddot{\varphi} = m_{red} \ddot{\varphi}$$

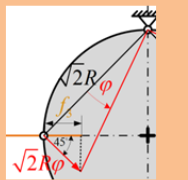
Q_c meghatározása

$$U = U_{12} + U_3 = \frac{1}{2} \frac{f_{12}^2}{c_1 + c_2} + \frac{1}{2} \frac{f_3^2}{c_3}$$

rugók hosszváltozása



$$f_{12} = \sqrt{2} R \cos 45^\circ = \sqrt{2} R \frac{\varphi}{2} = R \varphi$$



$$f_3 = \sqrt{2} R \cos 45^\circ = \sqrt{2} R \frac{\varphi}{2} = R \varphi$$

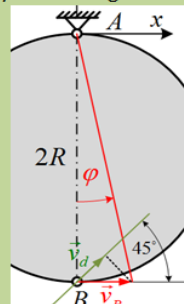
$$U = \frac{1}{2} \frac{(R \varphi)^2}{c_1 + c_2} + \frac{1}{2} \frac{(R \varphi)^2}{c_3} = \frac{1}{2} \left(\frac{R^2}{c_1 + c_2} + \frac{R^2}{c_3} \right) \varphi^2$$

$$Q_c = - \frac{\partial U}{\partial q} = - \frac{\partial U}{\partial \varphi} \frac{1}{2} \left(\frac{R^2}{c_1 + c_2} + \frac{R^2}{c_3} \right) 2 \varphi = - \left(\frac{R^2}{c_1 + c_2} + \frac{R^2}{c_3} \right) \varphi$$

$$Q_c = - \left(\frac{R^2}{c_1 + c_2} + \frac{R^2}{c_3} \right) \varphi = - \frac{1}{c_{red}} \varphi$$

Q_k meghatározása

dugattyú sebessége



$$v_B = 2R \dot{\varphi} \quad \vec{v}_B = 2R \dot{\varphi} \vec{i}$$

$$v_d = 2R \dot{\varphi} \cos 45^\circ = 2R \dot{\varphi} \frac{\sqrt{2}}{2} = \sqrt{2} R \dot{\varphi}$$

$$\vec{v}_d = v_d \vec{e}_d$$

$$\vec{v}_d = \sqrt{2} R \dot{\varphi} (\cos 45^\circ \vec{i} + \sin 45^\circ \vec{j})$$

$$\vec{v}_d = \sqrt{2} R \dot{\varphi} \left(\frac{\sqrt{2}}{2} \vec{i} + \frac{\sqrt{2}}{2} \vec{j} \right)$$

$$\vec{v}_d = R \dot{\varphi} \vec{i} + R \dot{\varphi} \vec{j}$$

$Q_k = \vec{F}_k \cdot \vec{\beta}_B$ ahol:

$$\vec{F}_k = -k \vec{v}_d = -k (R \dot{\varphi} \vec{i} + R \dot{\varphi} \vec{j})$$

$$\vec{\beta}_B = \frac{\partial \vec{v}_B}{\partial \dot{\varphi}} = \frac{\partial 2R \dot{\varphi} \vec{i}}{\partial \dot{\varphi}} = 2R \vec{i}$$

$$Q_k = \vec{F}_k \cdot \vec{\beta}_B = -k (R \dot{\varphi} \vec{i} + R \dot{\varphi} \vec{j}) 2R \vec{i} = -2k R^2 \dot{\varphi}$$

$$Q_k = -2R^2 k \dot{\varphi} = -k_{red} \dot{\varphi}$$

Q_g meghatározása

$Q_g = \vec{F}_g \cdot \vec{\beta}_B$ ahol:

$$\vec{F}_g = F_{g0} \sin(\omega t) \vec{i}$$

$$\vec{\beta}_B = \frac{\partial \vec{v}_B}{\partial \dot{\varphi}} = \frac{\partial 2R \dot{\varphi} \vec{i}}{\partial \dot{\varphi}} = 2R \vec{i}$$

$$Q_g = \vec{F}_g \cdot \vec{\beta}_B = F_{g0} \sin(\omega t) 2R \vec{i}$$

$$Q_g = 2R F_{g0} \sin(\omega t)$$

Numerikus eredmények :

$$m_{red} = \frac{3}{2} m R^2 = \frac{3}{2} \cdot 20 \cdot 0,5^2 = 7,5 \text{ kgm}^2$$

$$\frac{1}{c_{red}} = \frac{R^2}{c_1 + c_2} + \frac{R^2}{c_3} = \frac{0,5^2}{(2 + 4) \cdot 10^{-5}} + \frac{0,5^2}{5 \cdot 10^{-5}} = 9166,6 \text{ Nm}$$

$$k_{red} = 2R^2 k = 2 \cdot 0,5^2 \cdot 500 = 250 \text{ Nsm}$$

$$Q_g = 2R F_{g0} \sin(\omega t) = 2 \cdot 0,5 \cdot 200 \cdot \sin(10t) = 200 \cdot \sin(10t)$$

$$\frac{3}{2} m R^2 \ddot{\varphi}$$

+

$$0$$

$$- \left(\frac{R^2}{c_1 + c_2} + \frac{R^2}{c_3} \right) \varphi$$

+

$$-2R^2 k \dot{\varphi}$$

+

$$2R F_{g0} \sin(\omega t)$$

$$\frac{3}{2} m R^2 \ddot{\varphi}$$

+

$$2R^2 k \dot{\varphi}$$

+

$$\left(\frac{R^2}{c_1 + c_2} + \frac{R^2}{c_3} \right) \varphi$$

=

$$2R F_{g0} \sin(\omega t)$$

$$m_{red} \ddot{\varphi}$$

+

$$k_{red} \dot{\varphi}$$

+

$$\frac{1}{c_{red}} \varphi$$

=

$$2R F_{g0} \sin(\omega t)$$

$$7,5 \ddot{\varphi}$$

+

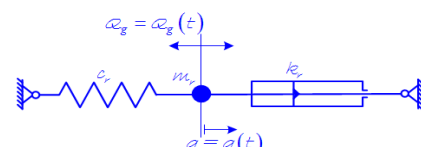
$$250 \dot{\varphi}$$

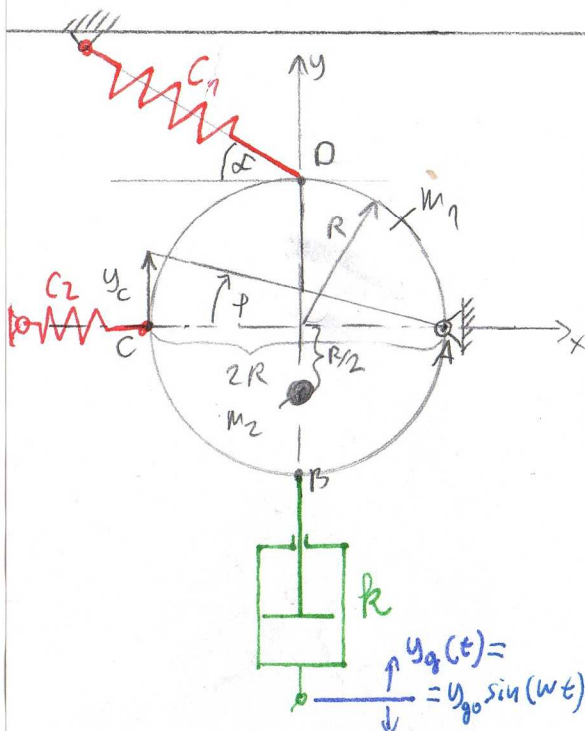
+

$$9166,6 \varphi$$

=

$$200 \sin(10t)$$





$m_1 = 20 \text{ kg}$
 $m_2 = 12 \text{ kg}$
 $R = 2 \text{ m}$
 $C_1 = 5 \cdot 10^4 \frac{\text{N}}{\text{m}}$
 $k = 1000 \frac{\text{N}}{\text{m}}$
 $y_{g0} = 15 \text{ mm}$
 $\omega = 50 \frac{\text{rad}}{\text{s}}$
 $\alpha = 30^\circ$

általános koordináta
 $q = y_c, \dot{q} = \dot{y}_c, \ddot{q} = \ddot{y}_c$
 $y_c = 2R\varphi \Rightarrow \varphi = \frac{y_c}{2R}$
 $\dot{y}_c = 2R\dot{\varphi} \Rightarrow \dot{\varphi} = \frac{\dot{y}_c}{2R}$
 $\ddot{y}_c = 2R\ddot{\varphi} \Rightarrow \ddot{\varphi} = \frac{\ddot{y}_c}{2R}$

Lagrange II. mozgásegyenlete

$$\frac{d}{dt} \left(\frac{\partial E}{\partial \dot{y}_c} \right) - \frac{\partial E}{\partial y_c} = Q_c + Q_k + Q_g$$

① ② ③ ④

a redukált vízgyővondó:

① $\frac{d}{dt} \left(\frac{\partial E}{\partial \dot{y}_c} \right)$ meghatározása:

$$\begin{aligned}
 E &= \frac{1}{2} J_O \omega^2 + \frac{1}{2} m_2 v_c^2 = \\
 &= \frac{1}{2} \left(\frac{3}{2} m_1 R^2 \right) \dot{\varphi}^2 + \frac{1}{2} m_2 \left(R^2 \dot{\varphi}^2 + \frac{R^2}{4} \dot{\varphi}^2 \right) \\
 &= \frac{1}{2} \left(\frac{3}{2} m_1 R^2 \right) \left(\frac{\dot{y}_c}{2R} \right)^2 + \frac{1}{2} m_2 \left(R^2 \frac{\dot{y}_c^2}{4R^2} + \frac{\dot{y}_c^2}{4R^2} \right) \\
 &= \frac{1}{2} \left(\frac{3}{2} m_1 R^2 \frac{1}{4R^2} + \left(R^2 + \frac{R^2}{4} \right) \frac{1}{4R^2} \right) \dot{y}_c^2 = \\
 &= \frac{1}{2} \left(\frac{3}{8} m_1 + \left(\frac{1}{4} + \frac{1}{16} \right) m_2 \right) \dot{y}_c^2 \\
 \frac{d}{dt} \frac{\partial E}{\partial \dot{y}_c} &= \left(\frac{3}{8} m_1 + \frac{5}{16} m_2 \right) \ddot{y}_c \\
 &= m_{red} \ddot{y}_c
 \end{aligned}$$

$$\begin{aligned}
 m_{red} &= \frac{3}{8} m_1 + \frac{5}{16} m_2 = \frac{3}{8} \cdot 20 + \frac{5}{16} \cdot 12 = \\
 &= 11,25 \text{ kg}
 \end{aligned}$$

② $\frac{\partial E}{\partial y_c} = 0$

visszahelyettesítve és átrendezve:

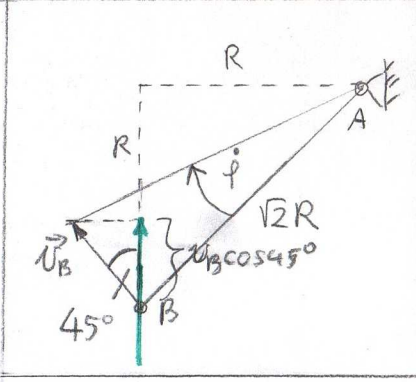
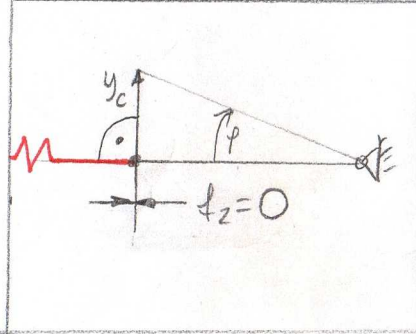
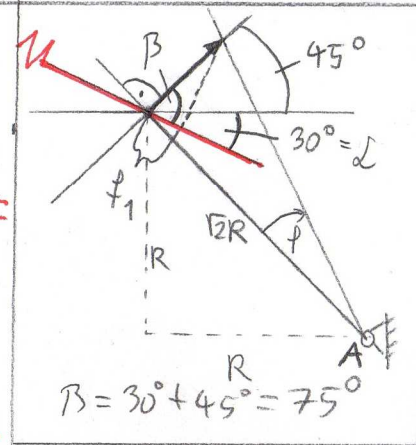
$$\begin{aligned}
 \left(\frac{3}{8} m_1 + \frac{5}{16} m_2 \right) \ddot{y}_c + \frac{1}{4} k \dot{y}_c + \frac{\cos^2 \beta}{2C_1} y_c &= \frac{1}{2} k y_{g0} \cos(\omega t) \\
 11,25 \ddot{y}_c + 250 \dot{y}_c + 67 y_c &= 375 \cos(50t) \\
 m_{red} \ddot{y}_c + k_{red} \dot{y}_c + \frac{1}{\cos \beta} y &= Q_g(t)
 \end{aligned}$$

③ Q_c meghatározása

$$\begin{aligned}
 U &= U_1 + U_2 = \frac{1}{2} \frac{f_1^2}{C_1} + \frac{1}{2} \frac{f_2^2}{C_2} = \\
 &= \frac{1}{2} \frac{(\sqrt{2} R \varphi \cos \beta)^2}{C_1} + \frac{1}{2} \frac{(2 R \varphi \cos 30^\circ)^2}{C_2} = \\
 &= \frac{1}{2} \frac{(\sqrt{2} R \frac{y_c}{2R} \cos \beta)^2}{C_1} + \frac{1}{2} \frac{0}{C_2} = \\
 &= \frac{1}{2} \left(\frac{1}{2} \frac{\cos^2 \beta}{C_1} \right) y_c^2 = \frac{1}{2} \left(\frac{\cos^2 \beta}{2C_1} \right) y_c^2 \\
 Q_c &= - \frac{\partial U}{\partial y_c} = - \frac{\cos^2 \beta}{2C_1} y_c \\
 \frac{1}{C_{red}} &= \frac{\cos^2 \beta}{2C_1} = \frac{\cos^2 75^\circ}{2 \cdot 5 \cdot 10^4} = 67 \frac{\text{N}}{\text{m}} \\
 C_{red} &= 67^{-1} = 0,015 \frac{\text{m}}{\text{N}}
 \end{aligned}$$

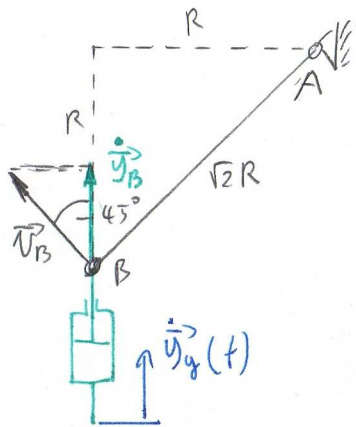
④ $Q_k + Q_g$ meghatározása

$$\begin{aligned}
 v_d &= \sqrt{2} R \dot{\varphi} \cos 45^\circ - \dot{y}_g(t) = \\
 &= \sqrt{2} R \frac{\dot{y}_c}{2R} \cdot \frac{\sqrt{2}}{2} - \dot{y}_g(t) = \frac{1}{2} \dot{y}_c - \dot{y}_g(t) \\
 D &= \frac{1}{2} k v_d^2 = \frac{1}{2} k \left(\frac{1}{2} \dot{y}_c - \dot{y}_g(t) \right)^2 \\
 Q_k + Q_g &= - \frac{\partial D}{\partial \dot{y}_c} = -k \left(\frac{1}{2} \dot{y}_c - \dot{y}_g(t) \right) \cdot \frac{1}{2} = \\
 &= - \frac{1}{4} k \dot{y}_c + \frac{1}{2} k \dot{y}_g(t) \\
 k_{red} &= \frac{1}{4} k = \frac{1}{4} \cdot 1000 = 250 \frac{\text{N}}{\text{m}}
 \end{aligned}$$



$$\begin{aligned}
 Q_g &= \frac{1}{2} k \omega \cdot y_{g0} \cos(\omega t) = \\
 &= \frac{1}{2} \cdot 1000 \cdot 50 \cdot 0,015 \cos(50t) = \\
 &= 375 \cos(50t) \text{ N}
 \end{aligned}$$

④ $Q_k + Q_y$ meghatározása (működő mód szer)



$$\begin{aligned} v_B &= \sqrt{2} R \dot{\theta} = \sqrt{2} R \frac{\dot{y}_c}{2R} = \frac{\sqrt{2}}{2} \dot{y}_c \\ \vec{v}_B &= v_B (-\sin 45^\circ \vec{i} + \cos 45^\circ \vec{j}) = \\ &= \frac{\sqrt{2}}{2} \dot{y}_c \left(-\frac{\sqrt{2}}{2} \vec{i} + \frac{\sqrt{2}}{2} \vec{j} \right) = \\ &= \underbrace{-\frac{1}{2} \dot{y}_c \vec{i}}_{\vec{x}_B} + \underbrace{\frac{1}{2} \dot{y}_c \vec{j}}_{\vec{y}_B} \\ \vec{v}_d &= (\vec{y}_B - \dot{y}_y(t)) \end{aligned}$$

$$\begin{aligned} \vec{v}_B &= \vec{v}_A + \vec{\omega} \times \vec{r}_{AB} = \vec{0} + (-\dot{\theta} \vec{h}) \times (-R \vec{i} - R \vec{j}) = \\ &= \dot{\theta} R (\vec{h} \times \vec{i}) + \dot{\theta} R (\vec{h} \times \vec{j}) = \\ &= -\dot{\theta} R \vec{j} + \dot{\theta} R \vec{i} = -\frac{\dot{y}_c}{2R} R \vec{i} + \frac{\dot{y}_c}{2R} R \vec{j} = \\ &= \underbrace{-\frac{1}{2} \dot{y}_c \vec{i}}_{\vec{x}_B} + \underbrace{\frac{1}{2} \dot{y}_c \vec{j}}_{\vec{y}_B} \\ \vec{v}_d &= (\vec{y}_B - \dot{y}_y(t)) \end{aligned}$$

$$Q_k = \vec{F}_h \cdot \vec{v}_B$$

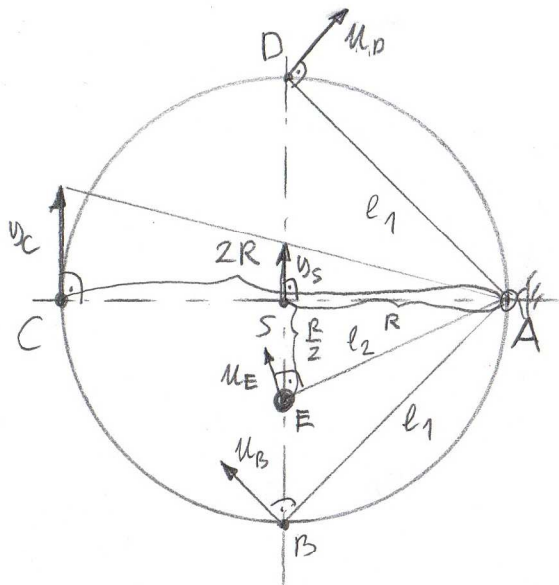
$$\begin{aligned} \vec{F}_h &= -h \vec{v}_d = -k (\vec{y}_B - \dot{y}_y(t)) = \\ &= -k \left(\frac{1}{2} \dot{y}_c \vec{j} - \dot{y}_y(t) \vec{j} \right) = \\ &= -k \cdot \left(\frac{1}{2} \dot{y}_c - \dot{y}_y(t) \right) \vec{j} \end{aligned}$$

$$\begin{aligned} \vec{v}_B &= \frac{\partial \vec{v}_B}{\partial \dot{y}_c} = \frac{\partial \left(-\frac{1}{2} \dot{y}_c \vec{i} + \frac{1}{2} \dot{y}_c \vec{j} \right)}{\partial \dot{y}_c} = \\ &= -\frac{1}{2} \vec{i} + \frac{1}{2} \vec{j} \end{aligned}$$

$$Q_k = \vec{F}_h \cdot \vec{v}_B = -k \left(\frac{1}{2} \dot{y}_c - \dot{y}_y(t) \right) \vec{j} \cdot \left(-\frac{1}{2} \vec{i} + \frac{1}{2} \vec{j} \right) = \underbrace{-\frac{1}{2} k \dot{y}_c}_{Q_k} + \underbrace{\frac{1}{2} k \dot{y}_y(t)}_{Q_y}$$

megjegyzés: f nélkül arányosság alakján:

A forgyásponthól távolodva az elemzővel (és annak deriváltjai) lineárisan nőnek



alt koord: y_c

$$y_s = \frac{R}{2R} y_c = \frac{1}{2} y_c$$

$$u_D = \frac{e_1}{2R} y_c = \frac{\sqrt{2}R}{2R} y_c = \frac{\sqrt{2}}{2} y_c \Rightarrow V = \frac{1}{2} \frac{u_D^2}{c_1} = \frac{1}{2} \frac{(u_D \cos \beta)^2}{c_1} = \frac{1}{2} \frac{\left(\frac{\sqrt{2}}{2} y_c \cos \beta\right)^2}{c_1} = \frac{1}{2} \left(\frac{\frac{1}{2} \cos^2 \beta}{c_1}\right) y_c^2$$

$$v_B = \dot{u}_B = \frac{\ell_1}{2R} \dot{y}_c = \frac{\sqrt{2}R}{2R} \dot{y}_c = \frac{\sqrt{2}}{2} \dot{y}_c \Rightarrow D = \frac{1}{2} k (\dot{u}_B \cos 45^\circ - \dot{y}_g(t))^2 = \frac{1}{2} k \left(\frac{\sqrt{2}}{2} \dot{y}_c \frac{\sqrt{2}}{2} - \dot{y}_g(t) \right)^2 = \frac{1}{2} k \left(\frac{1}{2} \dot{y}_c - \dot{y}_g(t) \right)^2$$

$$v_2 = v_E = \dot{u}_E = \frac{l_2}{2R} \dot{y}_c = \frac{\sqrt{R^2 + \frac{R^2}{4}}}{2R} \dot{y}_c = \sqrt{\frac{R^2 + \frac{R^2}{4}}{4R^2}} \dot{y}_c = \sqrt{\frac{1}{4} + \frac{1}{16}} \dot{y}_c = \sqrt{\frac{5}{16}} \dot{y}_c$$

$$\Rightarrow E_2 = \frac{1}{2} m_2 v_2^2 = \frac{1}{2} m_2 \left(\sqrt{\frac{5}{16}} \dot{y}_c \right)^2 = \frac{1}{2} \left(\frac{5}{16} m_2 \right) \dot{y}_c^2$$